



AMCA International

# Ascertaining Efficiency For Air-Side Equipment

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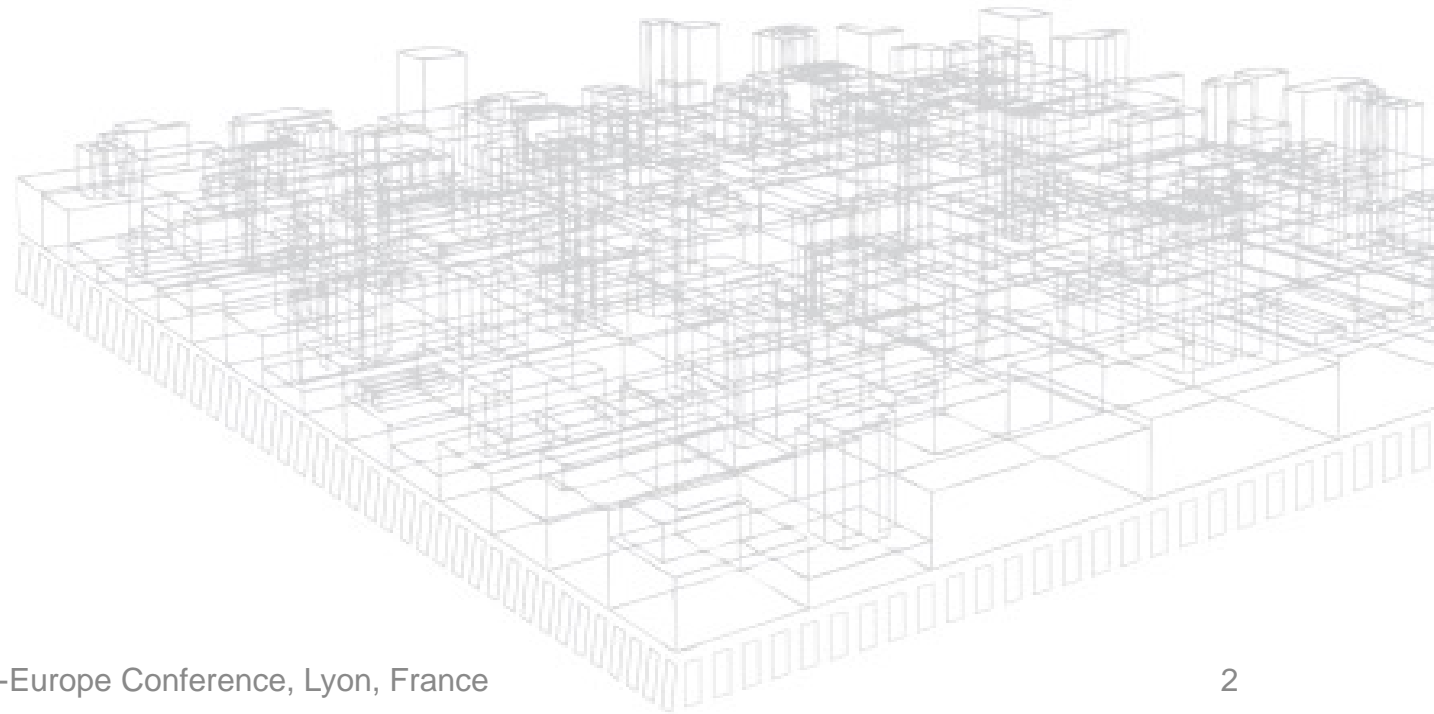
**Air System Engineering & Technology (ASET) Conference-Europe**

**Lyon, France • L'Espace Tête d'Or • 20 February 2018**

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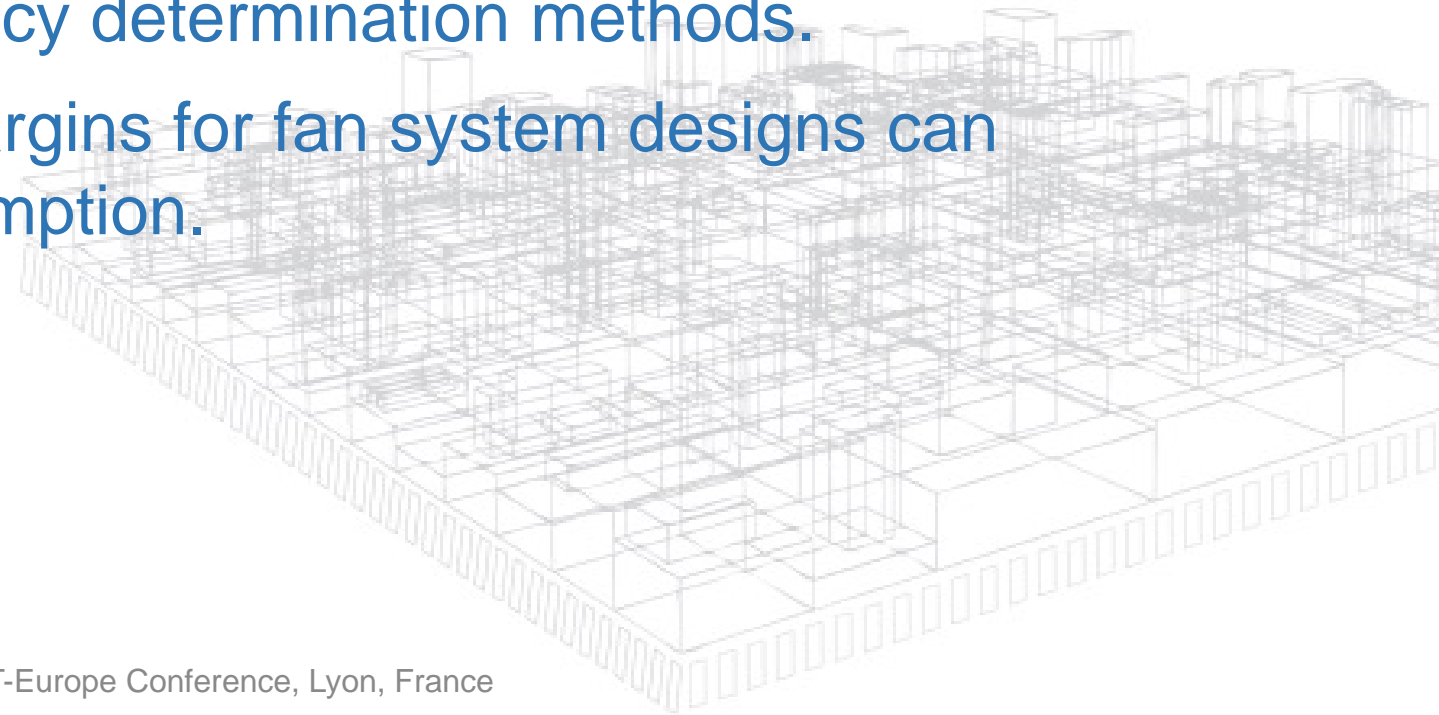
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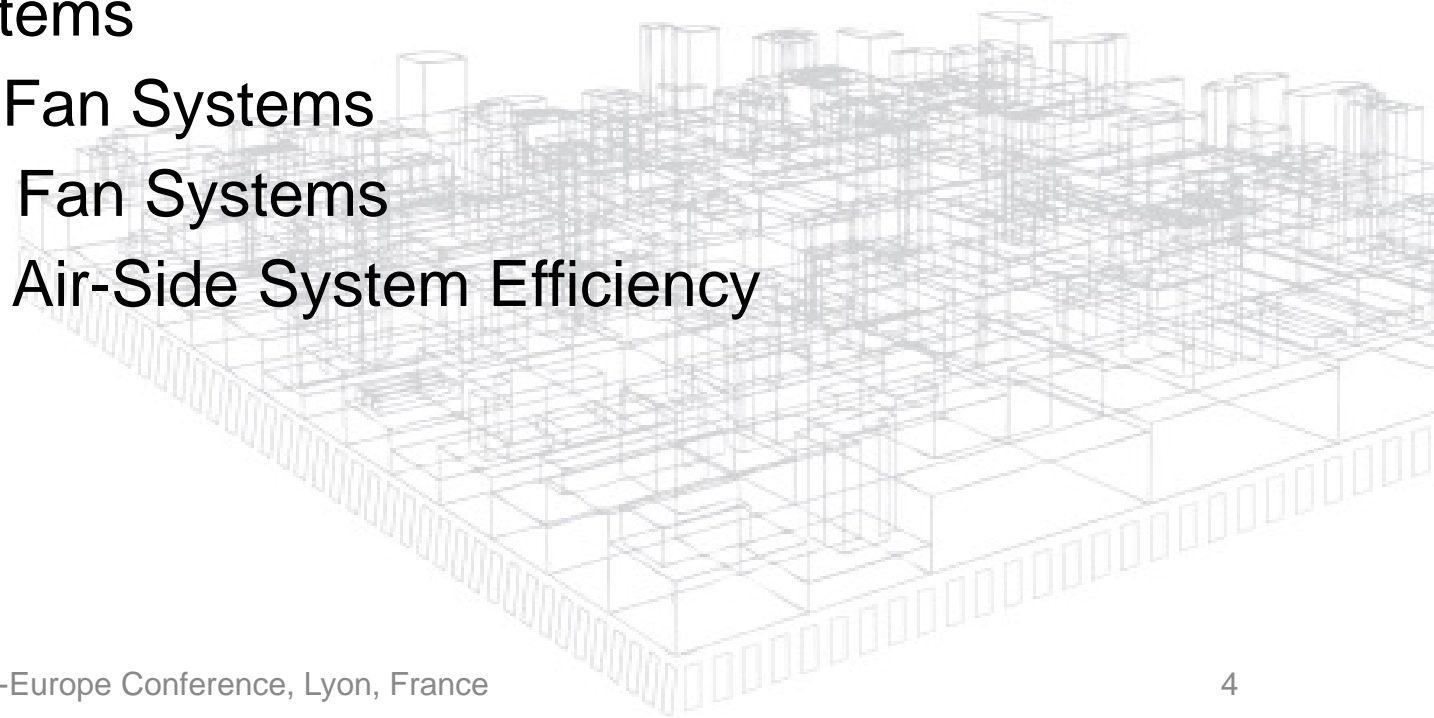
# Learning Objectives

- Introduce a new performance-based fan energy efficiency metric.
- Compare fan efficiency determination methods.
- Show how safety margins for fan system designs can impact power consumption.



# Agenda

- Introduction of the Fan Energy Index
- Air Side Equipment
- Efficiency of Air-Side Systems
- Component Based Fan Systems
- Fully Integrated and Tested Fan Systems
- Efficiency Characteristics of Fan Systems
- Effects of Safety Factors on Air-Side System Efficiency
- Final Notes



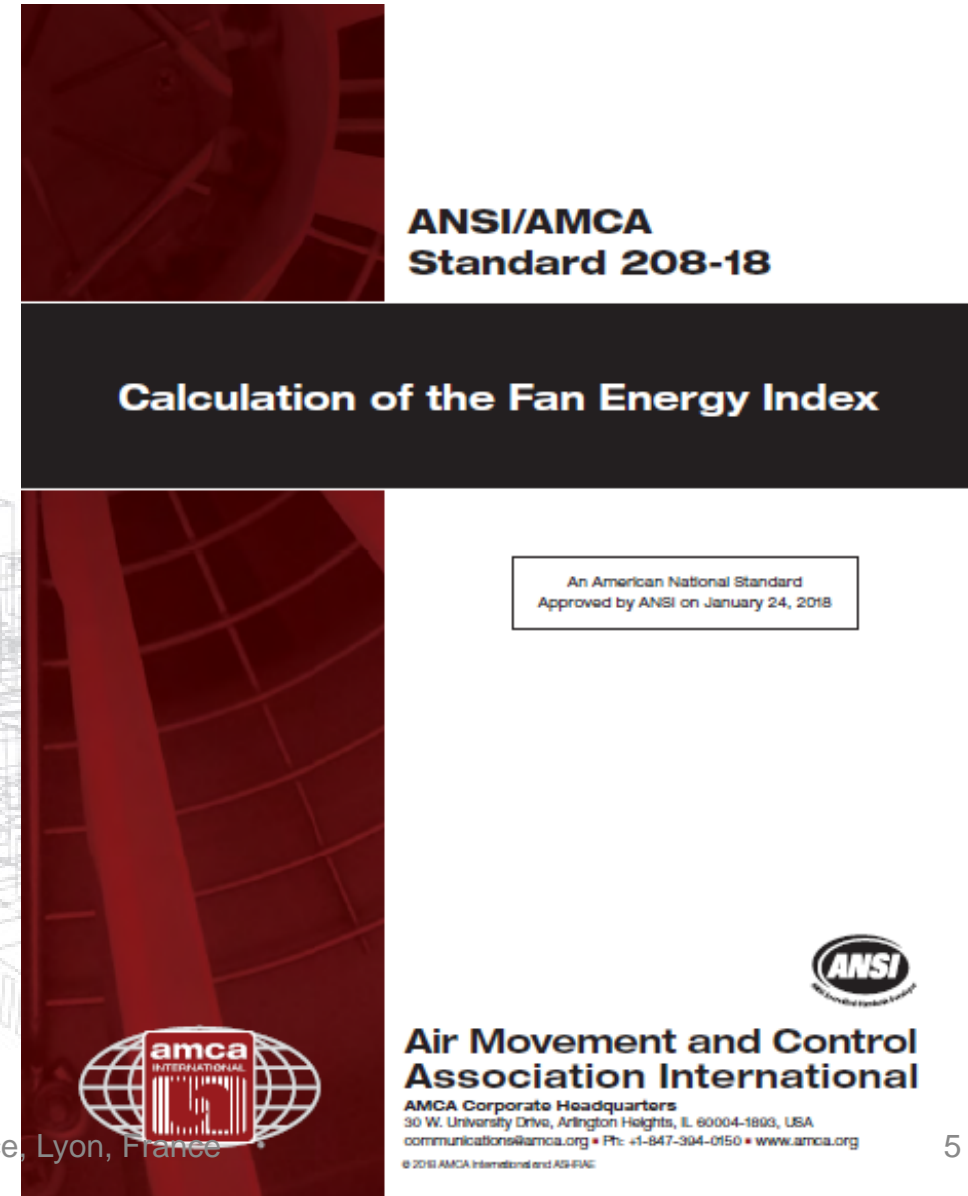
# Fan Energy Index

## ANSI/AMCA Standard 208

- Calculation method for the fan energy index (FEI)
- Energy efficiency metric for fans includes motor and drive losses.
- Individual calculation at each given fan duty point.
- FEI combines fan performance and efficiency in one value.

### Uses:

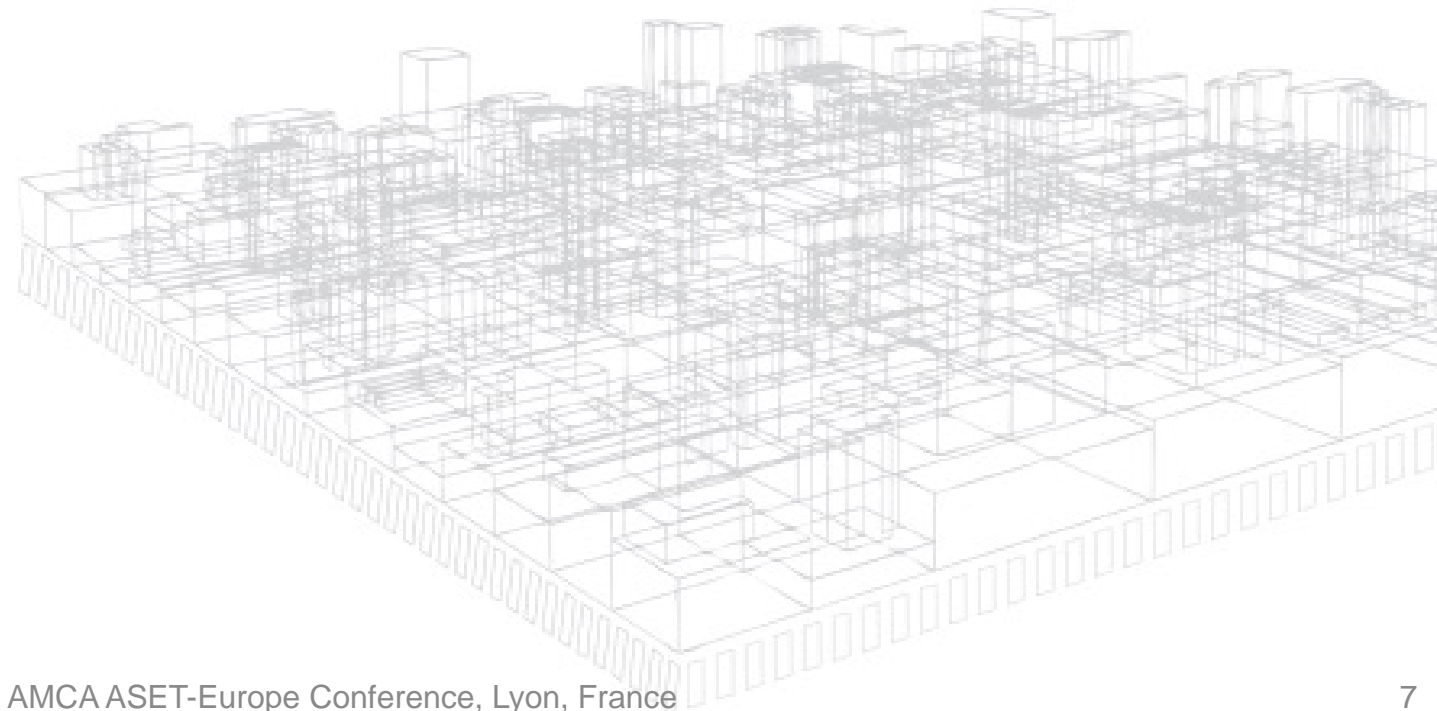
- Standardized and consistent **comparison** of fans across fan types and sizes.
- **Definition of** the energy **requirements** of fans.



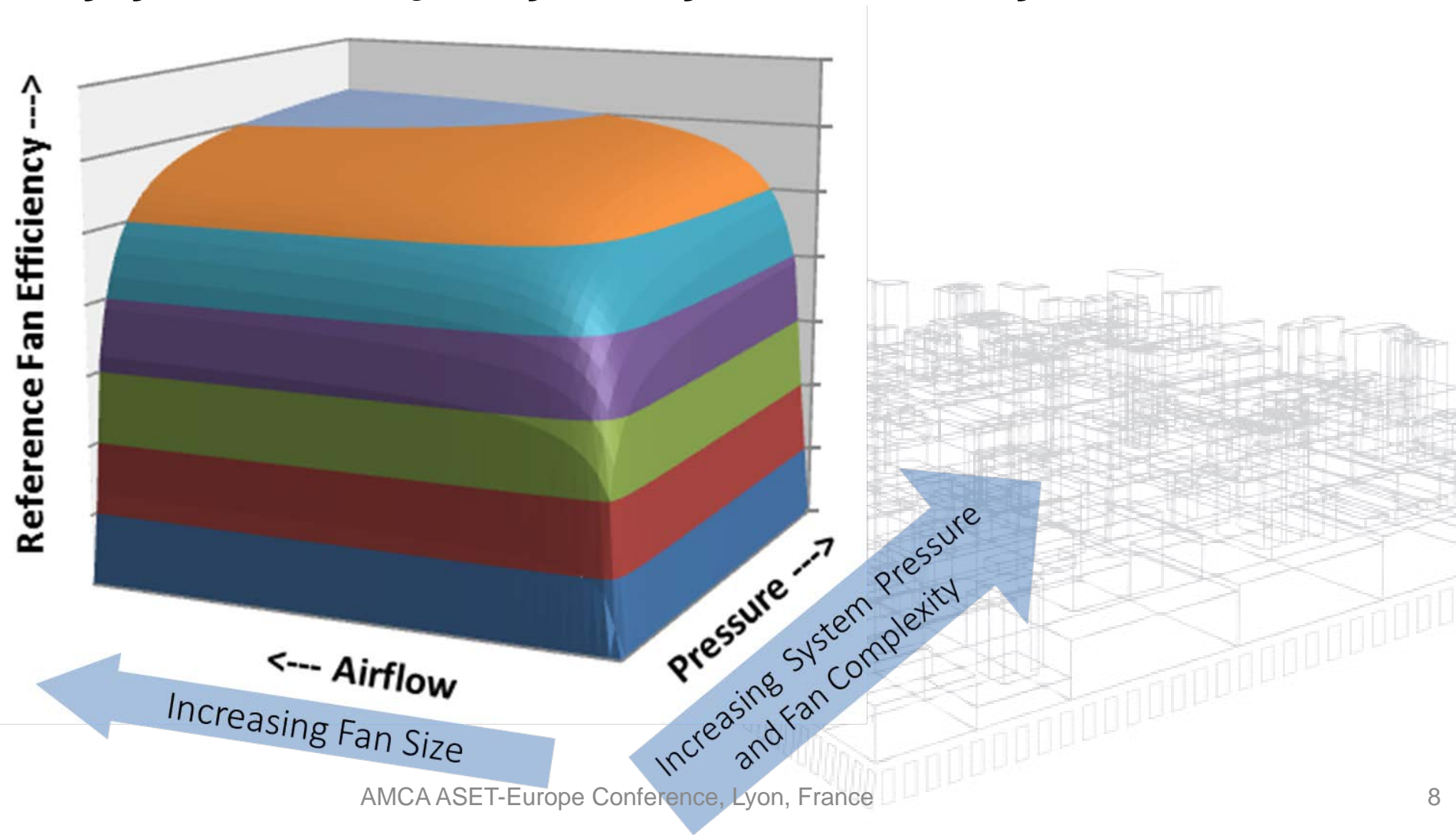
Efficiency metric	Criterion	Rating point	Use
Ecodesign 327/2011 or FMEG Fan Motor Efficiency Grade	Wire-to-air efficiency & Fan type	Best efficiency point	How good is the fan?
FEG Fan Efficiency Grade	Peak Total Efficiency & Diameter	Best efficiency point	How good is the bare-shaft fan?
FEI Fan Energy Index	Electric input power	Selected fan duty point	How good is the fan for its application?

# Fan Energy Index ANSI/AMCA Standard 208

FEI is evaluated at every operating point offered for sale.

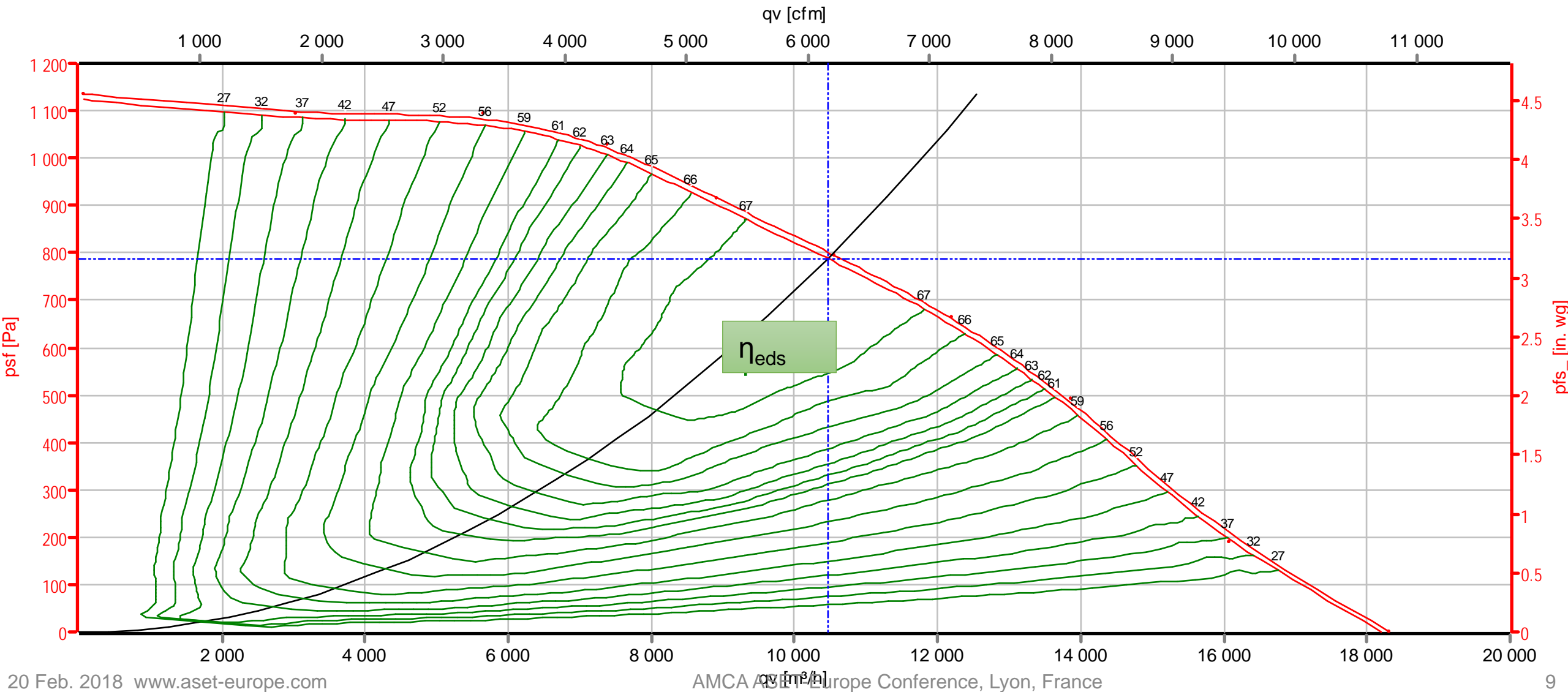


$$FEI = \frac{\text{efficiency of selected fan}}{\text{efficiency of reference fan}}$$

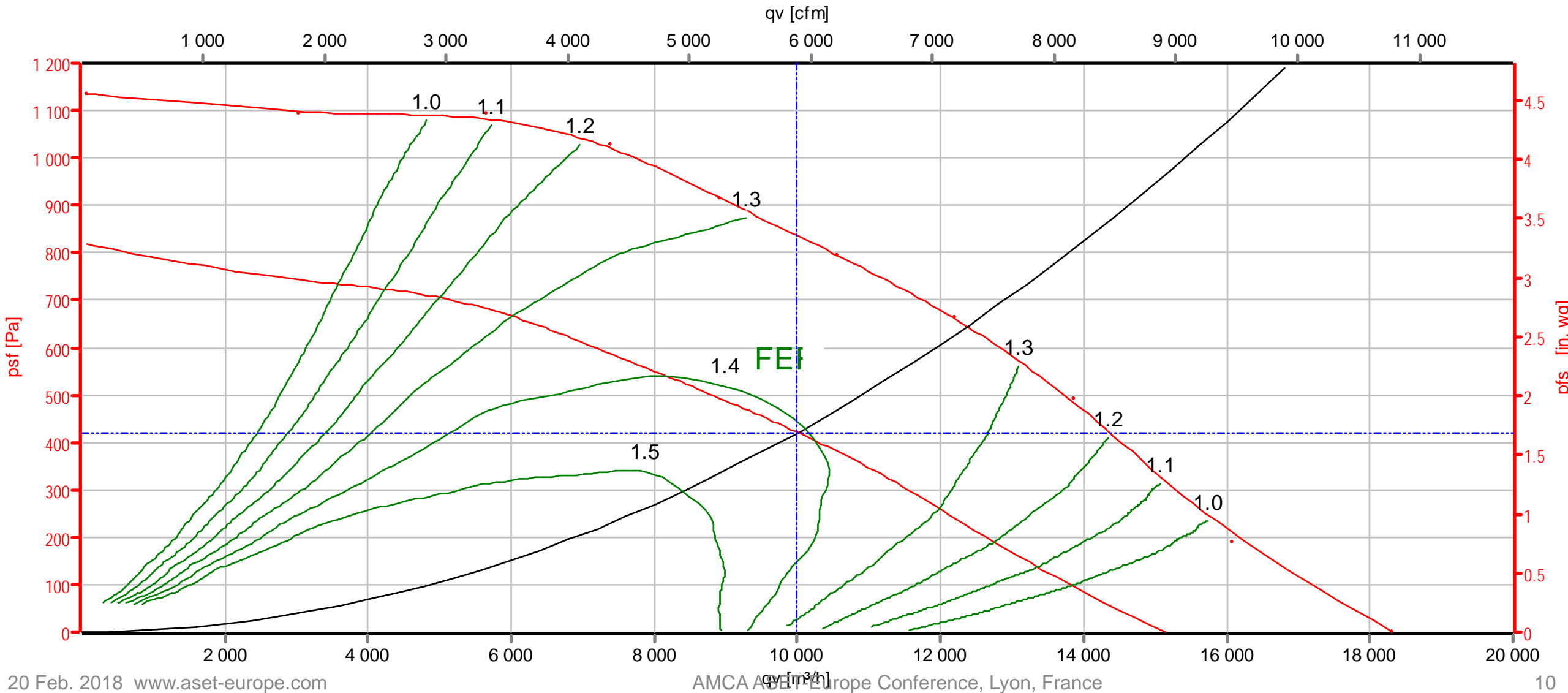




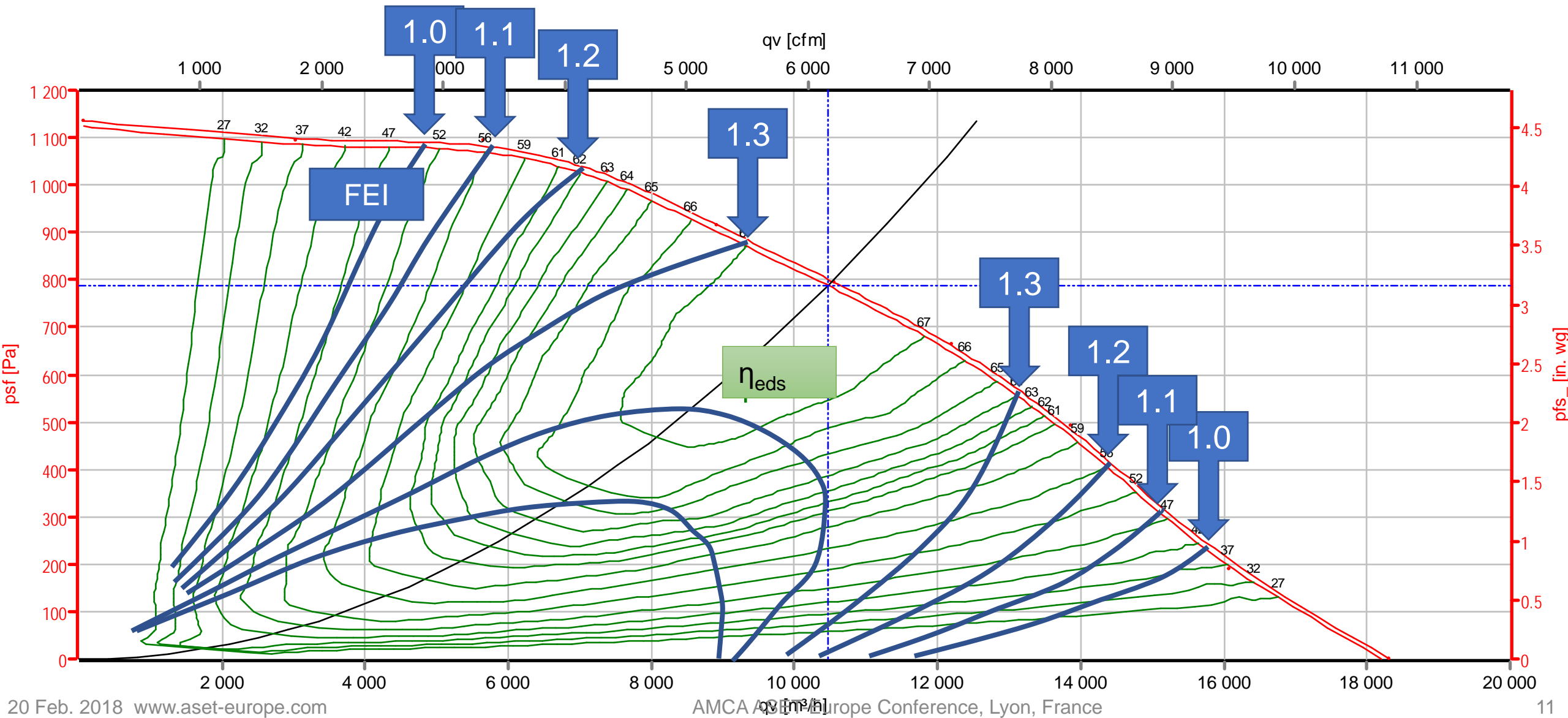
# Lines of constant fan efficiency



# Lines of constant FEI



# Lines of constant FEI

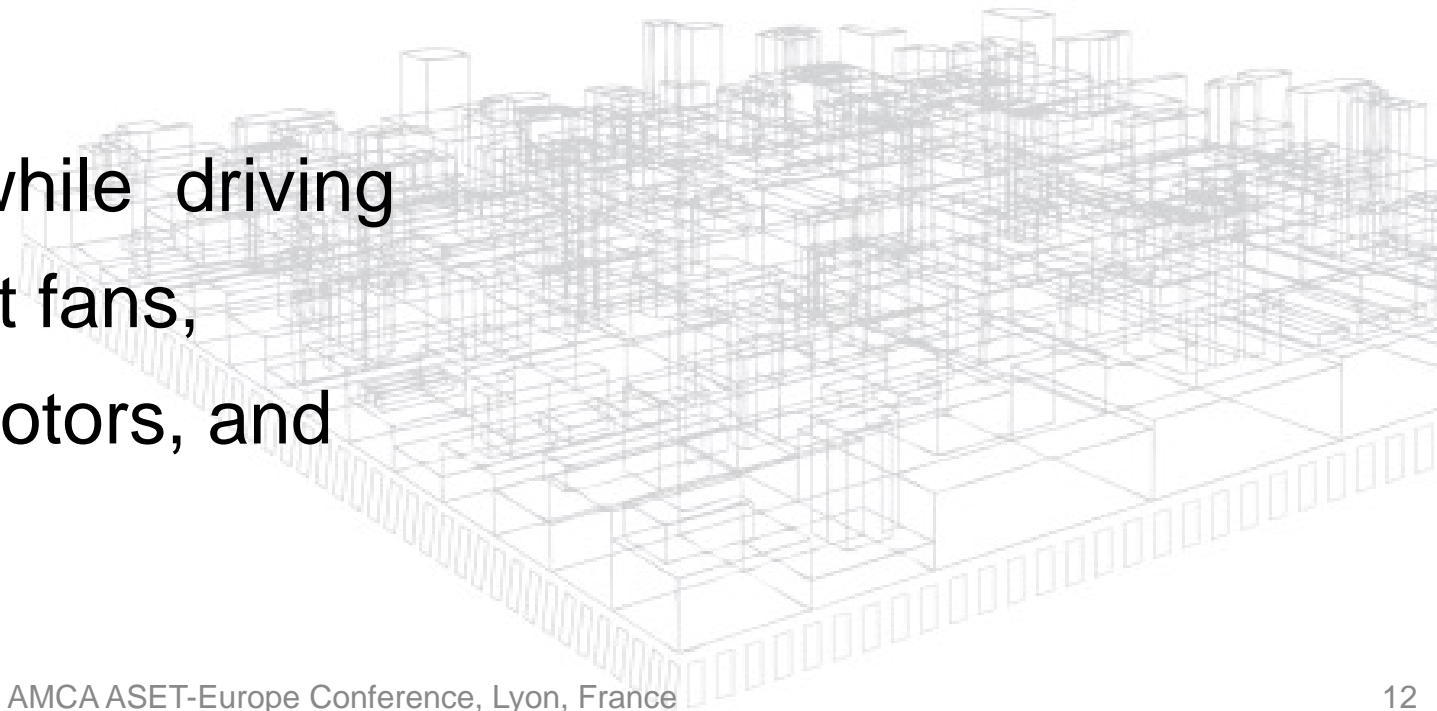


# Fan Energy Index summary

Fan **peak** efficiency metrics are barely effective if a fan may be selected at a low **operational** efficiency.

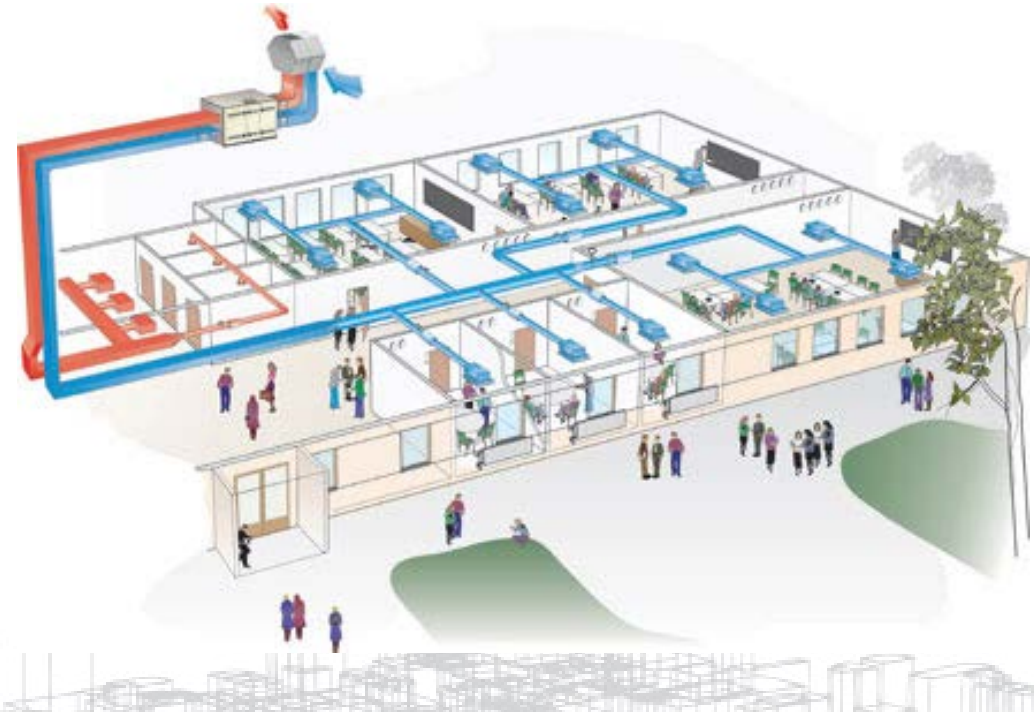
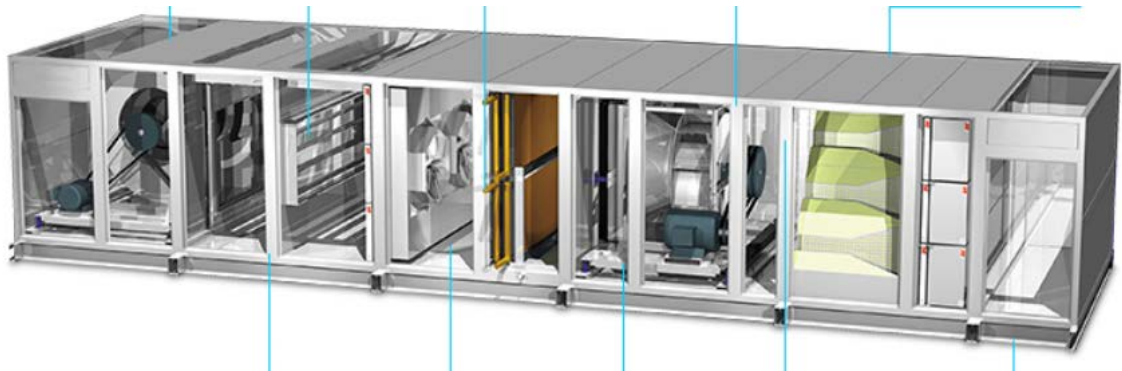
FEI will encourage

- more efficient selections, while driving
- the design of more efficient fans,
- the use of more efficient motors, and
- the use of direct drives.



# Air-Side Equipment

Provides defined amounts of treated air to the spaces, people, or processes.

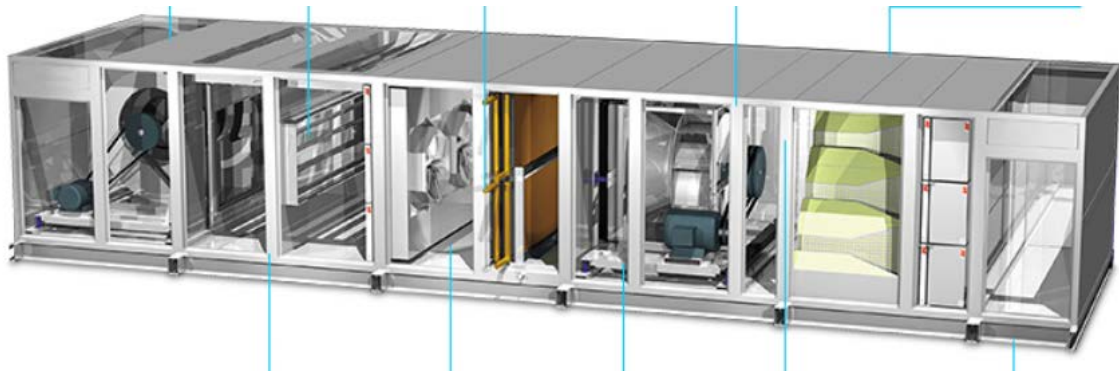
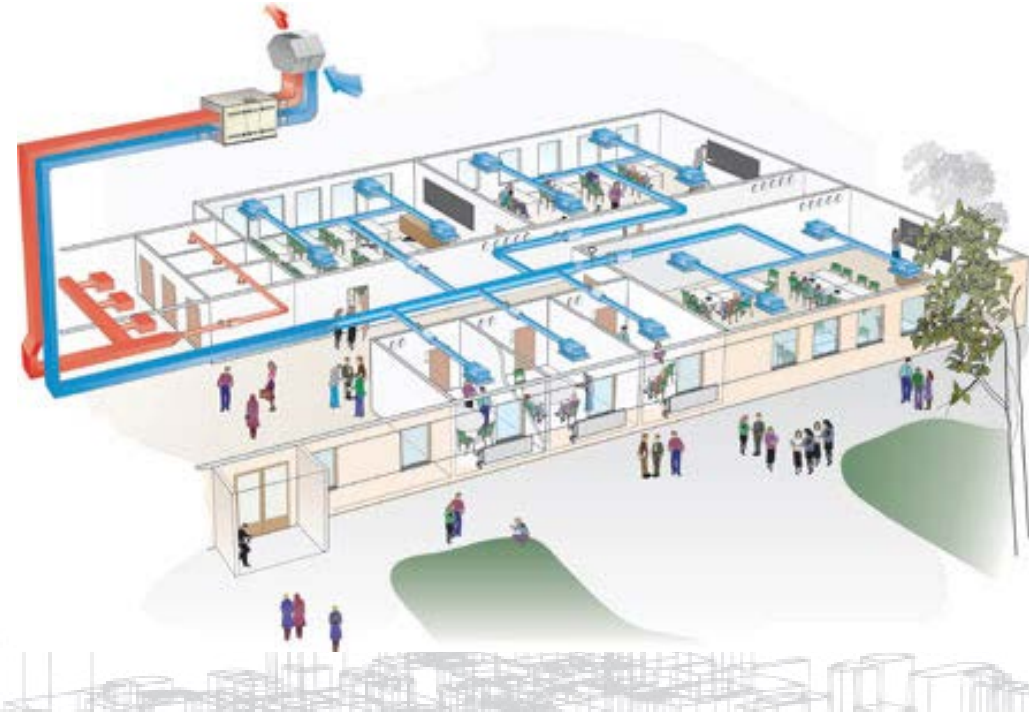


Includes air distribution components and fan-powered units.

- Air handling units (AHU)
- Fan coil units (FCU)
- Roof-top units (RTU)

# Air-Side Equipment

The air volume requirement is defined by the heat load, cooling load, or ventilation rate for health and comfort.

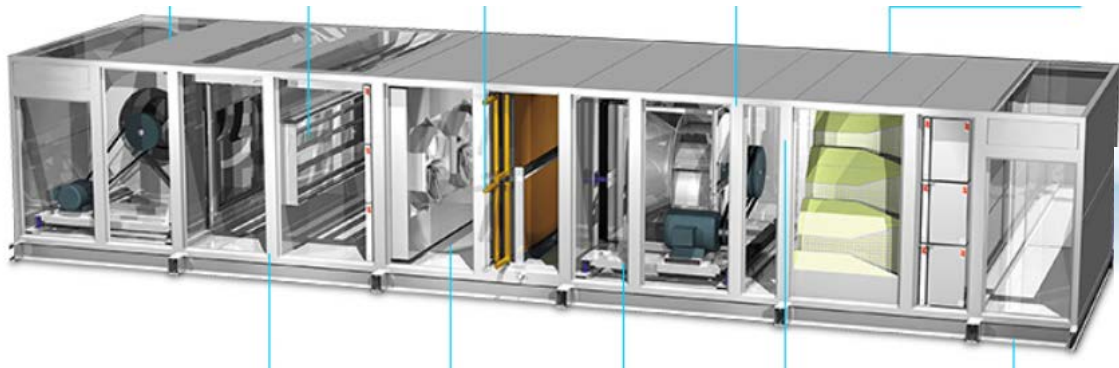


The system that can fulfil all the requirements with the least amount of energy is the most efficient.



# Air-Side Equipment

- Air distribution system components cause external pressure losses.



- Air handling unit components cause internal pressure losses.

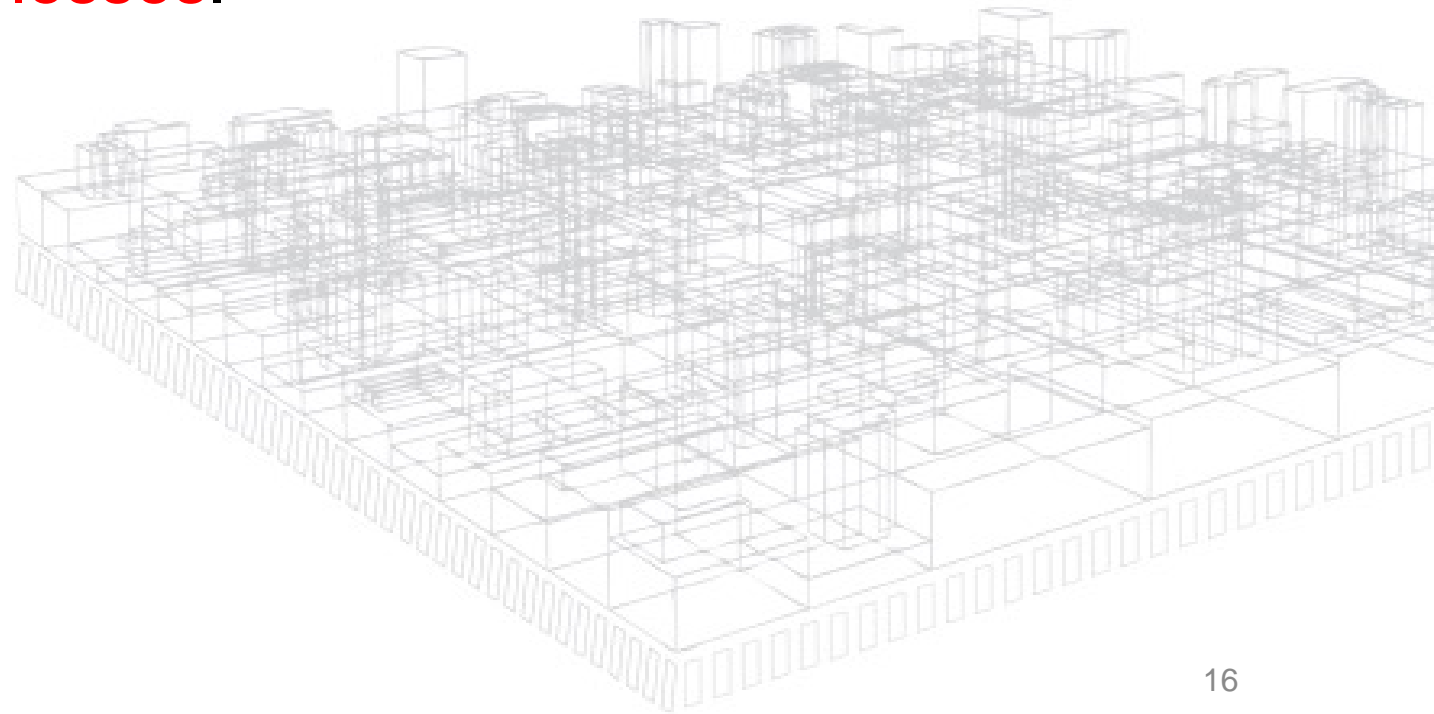
# Air-Side Equipment

Defined **amounts of** treated **air** to the spaces, people, or processes.

$$q_v \times p = Pu$$

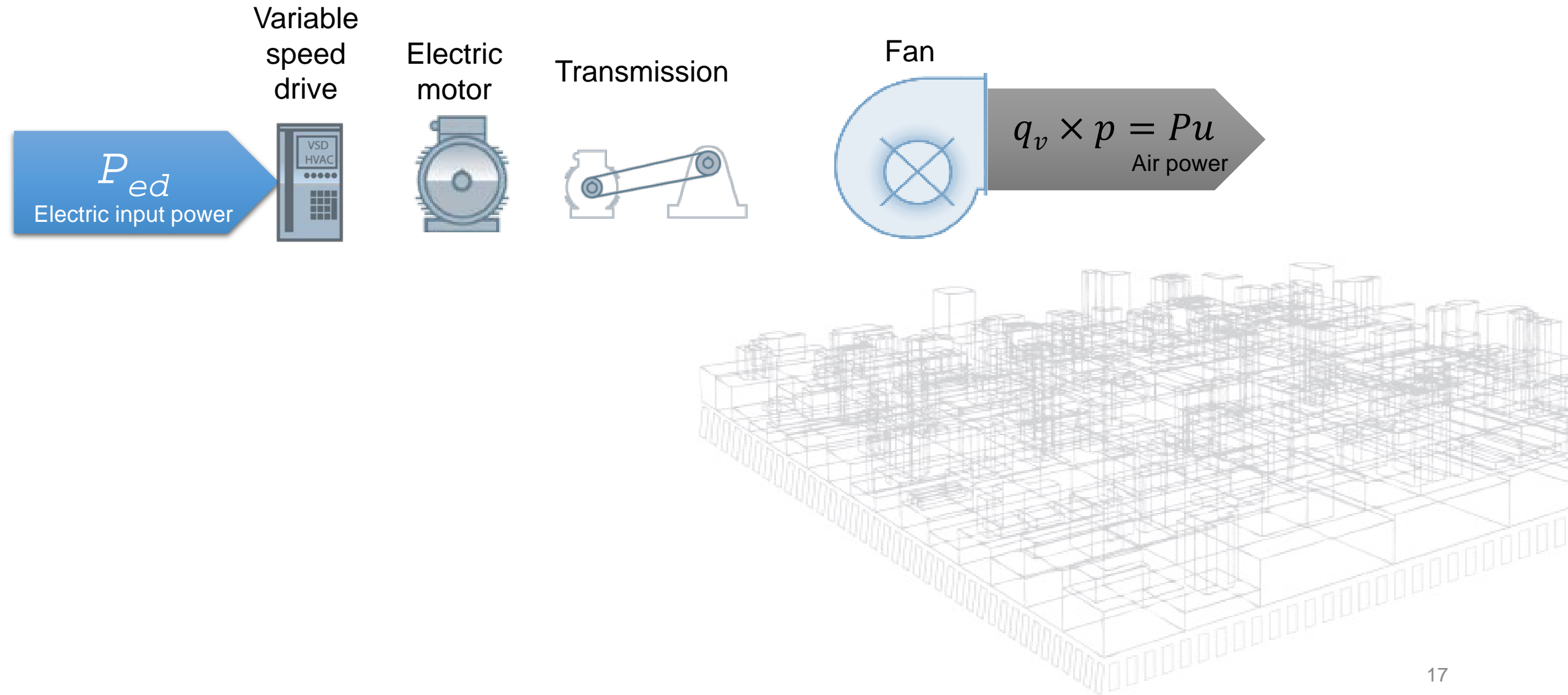
Air power

Internal and external **pressure losses**.

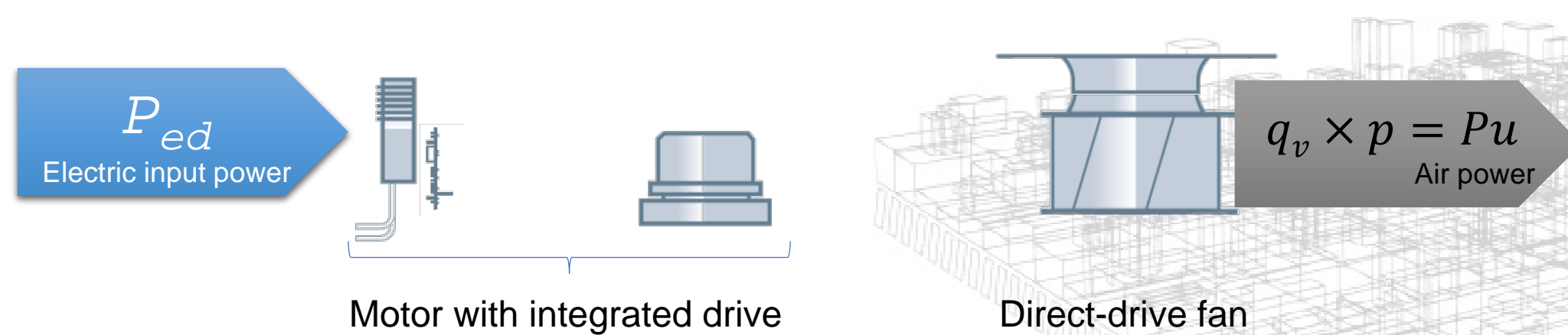
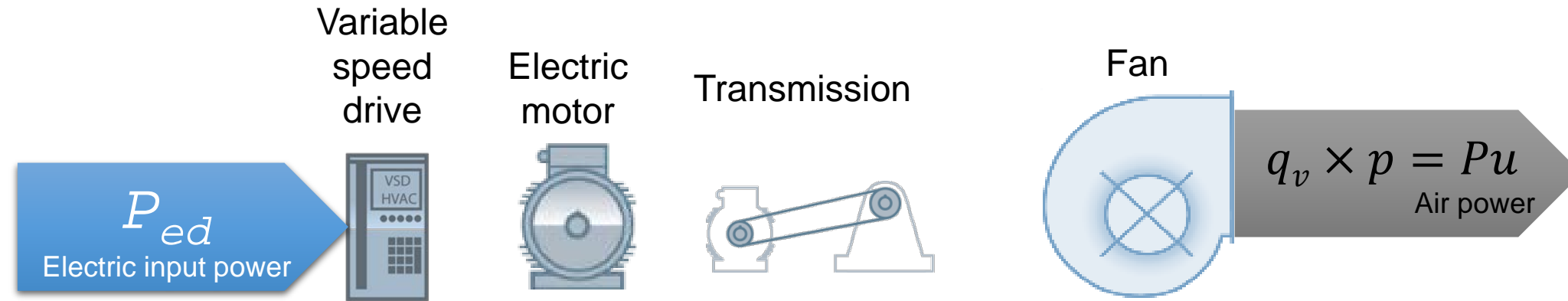




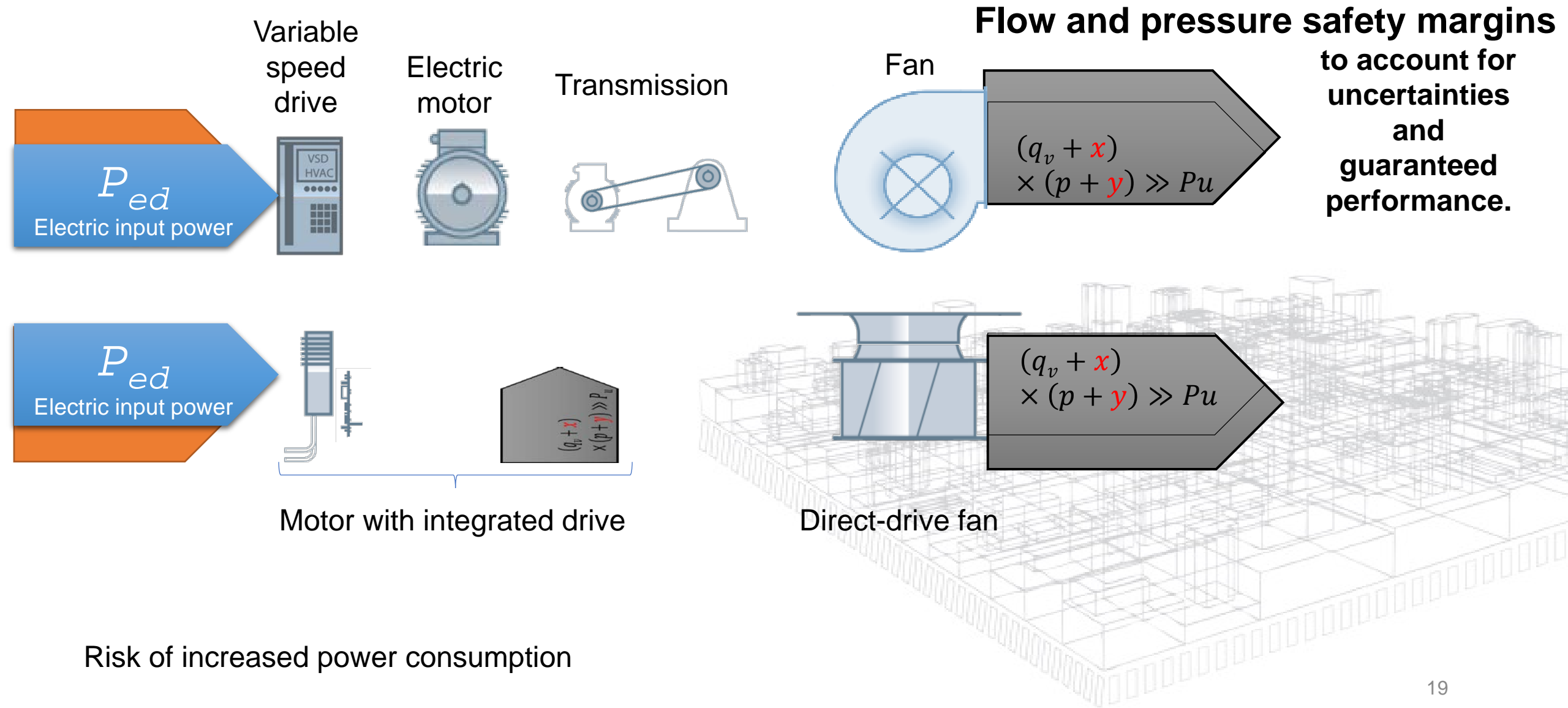
# Efficiency of Air-Side Systems



# Efficiency of Air-Side Systems

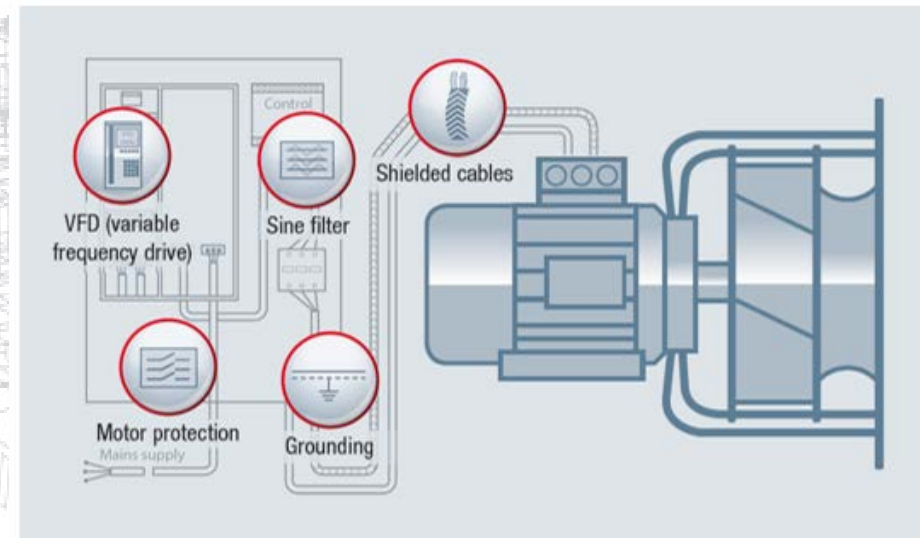
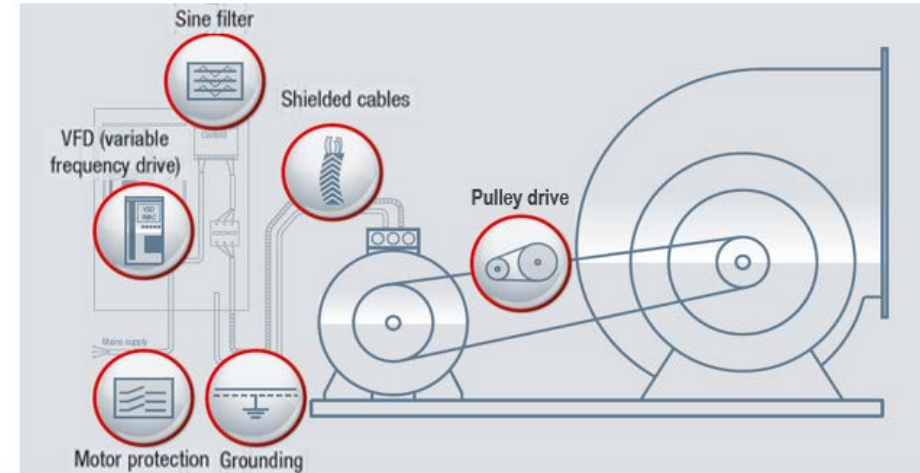


# Efficiency of Air-Side Systems

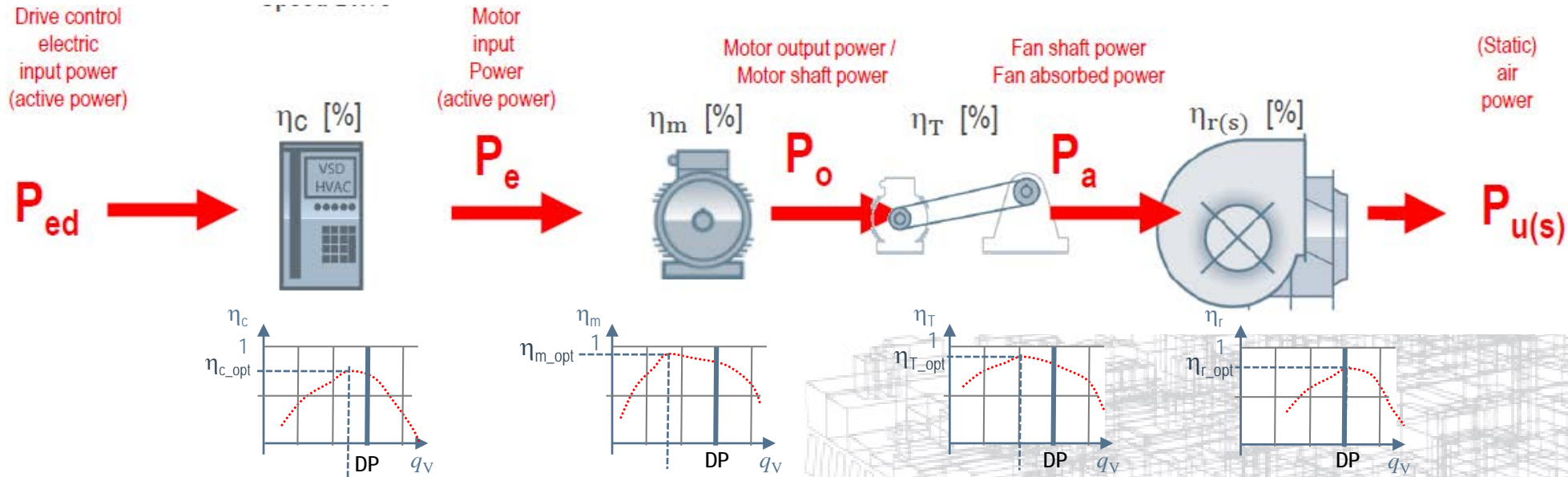


# Component-Based Fan Systems

- Each component is tested by itself, not as an entire system.
- Individual datasheets and testing standards for assessing performance and characteristic.
- The contractor is responsible for compliance with relevant standards.
- Component sourcing may be centralized in the project execution or split between contractors (e.g. Variable Frequency Drive, filters, cables) and the OEM (fan impeller, motor, transmission).



# Component-Based Fan Systems



$\eta_{c\_max}$

✗

$\eta_{m\_max}$

✗

$\eta_{T\_max}$

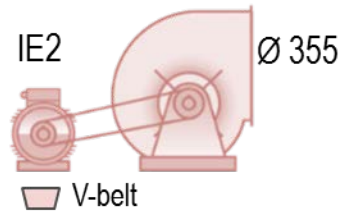
✗

$\eta_{rs\_max}$

≠

$\eta_{es}$


# Component-Based Fan Systems



Fan duty point:  $q_V = 8,800 \text{ m}^3/\text{h}$  @  $p_f = 1,100 \text{ Pa}$

	AMCA 207	ISO 12759	Wire-to-air test
Fan total efficiency in %			
Impeller shaft power in kW			
Belt efficiency in %			
Motor shaft power in kW			
Motor efficiency in %			
Wire-to-air efficiency in %			
Motor input power in kW			

# Component-Based Fan Systems




STANDARD

**ANSI/AMCA  
Standard 207-17**

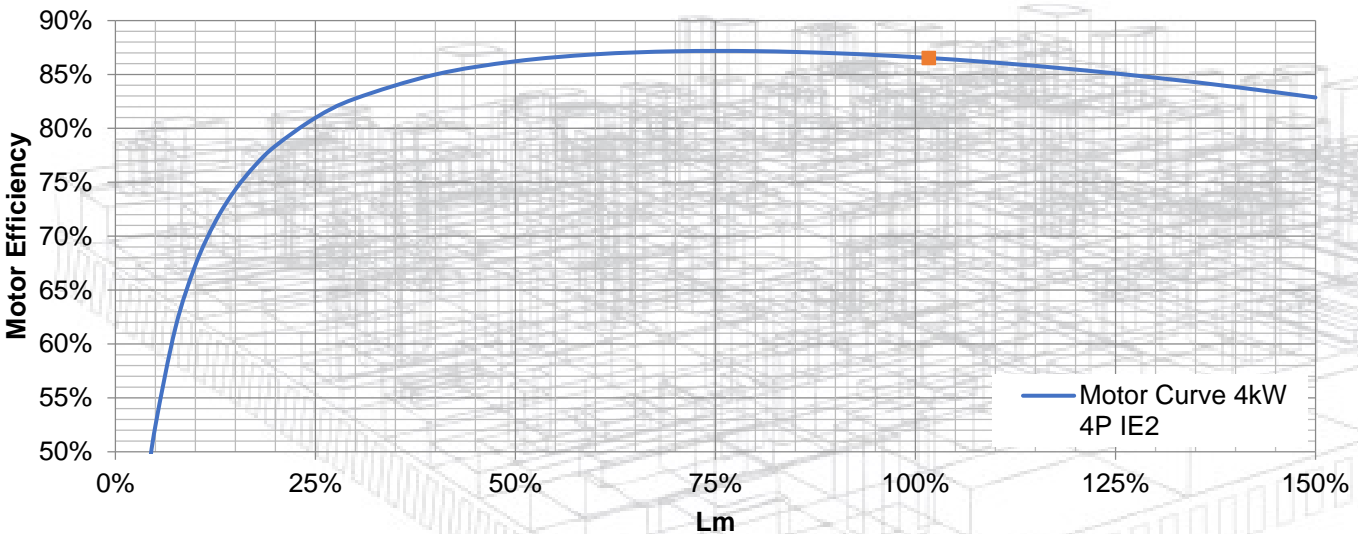
**Fan System Efficiency and  
Fan System Input Power Calculation**

An American National Standard  
Approved by ANSI on April 17, 2017

 **Air Movement and Control  
Association International**

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Motor Selector			Transmission	Fan Data				
Motor Power (kW)	Class	Poles		Q (m <sup>3</sup> /s)	Fan P (Pa)	Fan Eff	N	H (kW)
4	IE2	4	ISO V-Belt	2.44	1182	75.7%	1700	3.8
Nom Efficiency	Motor Ns							
86.6%	1500							
Motor Output		Component Efficiencies			Net kW in			
Load Ratio	Power (kW)	Belt ETA	Motor ETA	Net ETA				
101.7%	4.1	93.9%	86.5%	61.5%	4.70			





# Component-Based Fan Systems

## ISO 12759:2010 Fans – Efficiency classification for fans

If fan drives are sold as components and combined without measurement then this standard permits multiplying the component efficiency maxima as follows:

$$\eta_e = \eta_r \times \eta_m \times \eta_T \times \eta_c \times C_m \times C_c$$

where

$\eta_e$  is the overall efficiency;

$\eta_r$  is the optimal fan impeller efficiency according to  $P_{u(s)}/P_a$ , as given in ISO 5801;

$\eta_m$  is the motor efficiency;

$\eta_T$  is the drive mechanism (transmission efficiency);

$\eta_c$  is the variable speed drive efficiency;

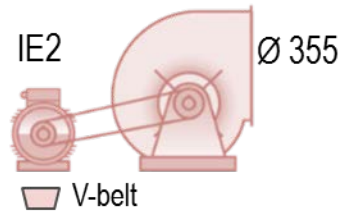
$C_m$  is the compensation factor to account for matching of components = 0,9;

$C_c$  is part load compensation factor.

Standardized compensation to account for mismatched component efficiency curves.



# Component-Based Fan Systems



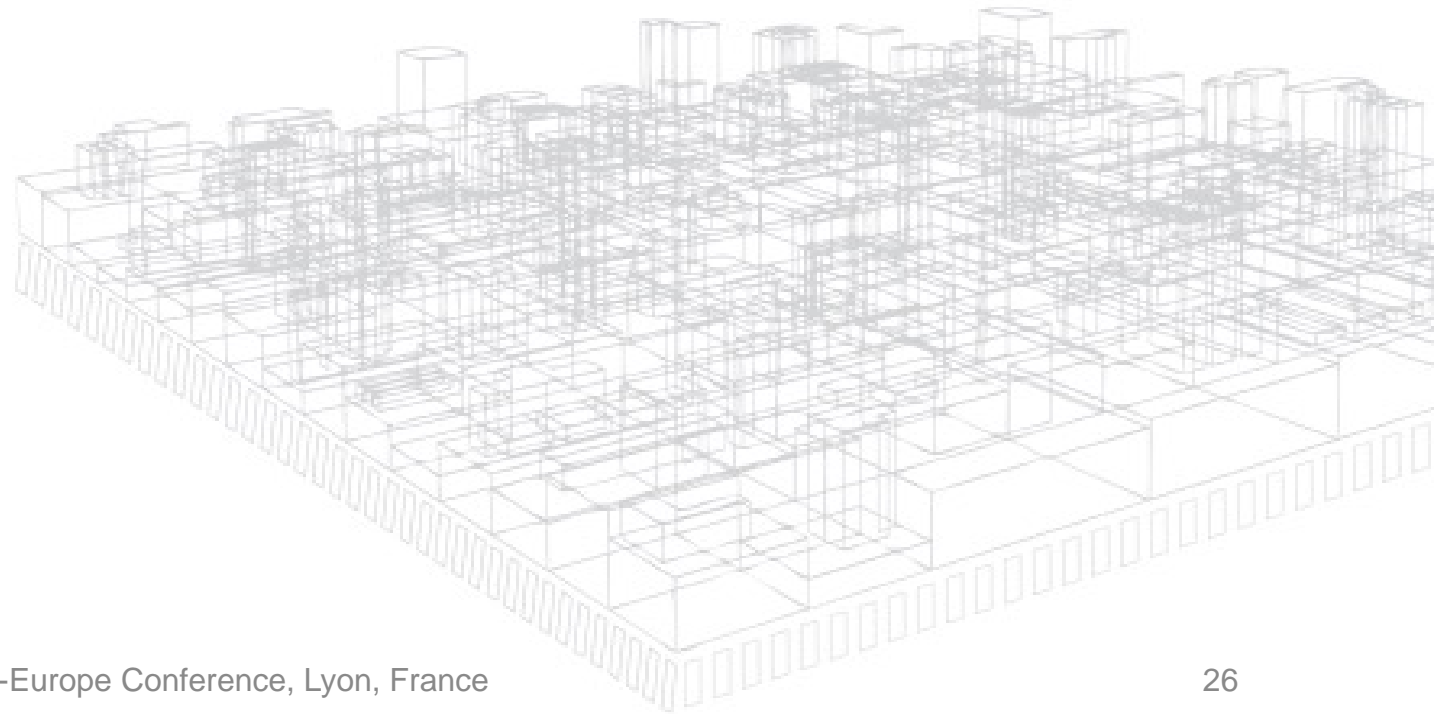
Fan duty point:  $q_V = 8,800 \text{ m}^3/\text{h}$  @  $p_f = 1,100 \text{ Pa}$

	AMCA 207	ISO 12759	Wire-to-air test
Fan total efficiency in %	75.7	75.7	
Impeller shaft power in kW	3.82	3.82	
Belt efficiency in %	93.9	94.3	
Motor shaft power in kW	4.10	4.05	
Motor efficiency in %	86.5	86.6	
Wire-to-air efficiency in %	61.5	55.6 incl. $C_m$	
Motor input power in kW	4.7	5.2	

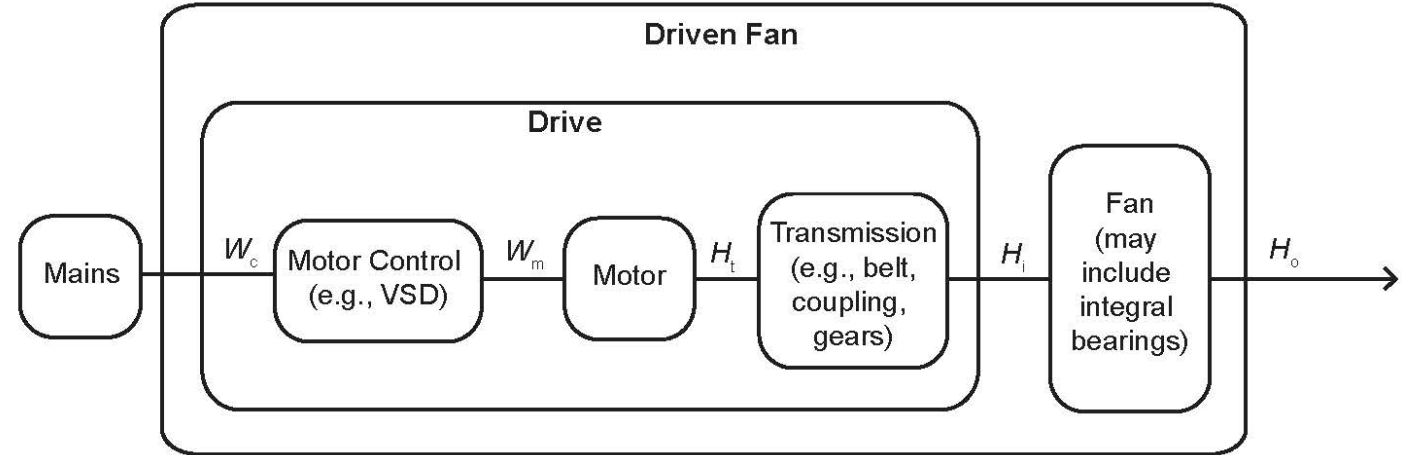
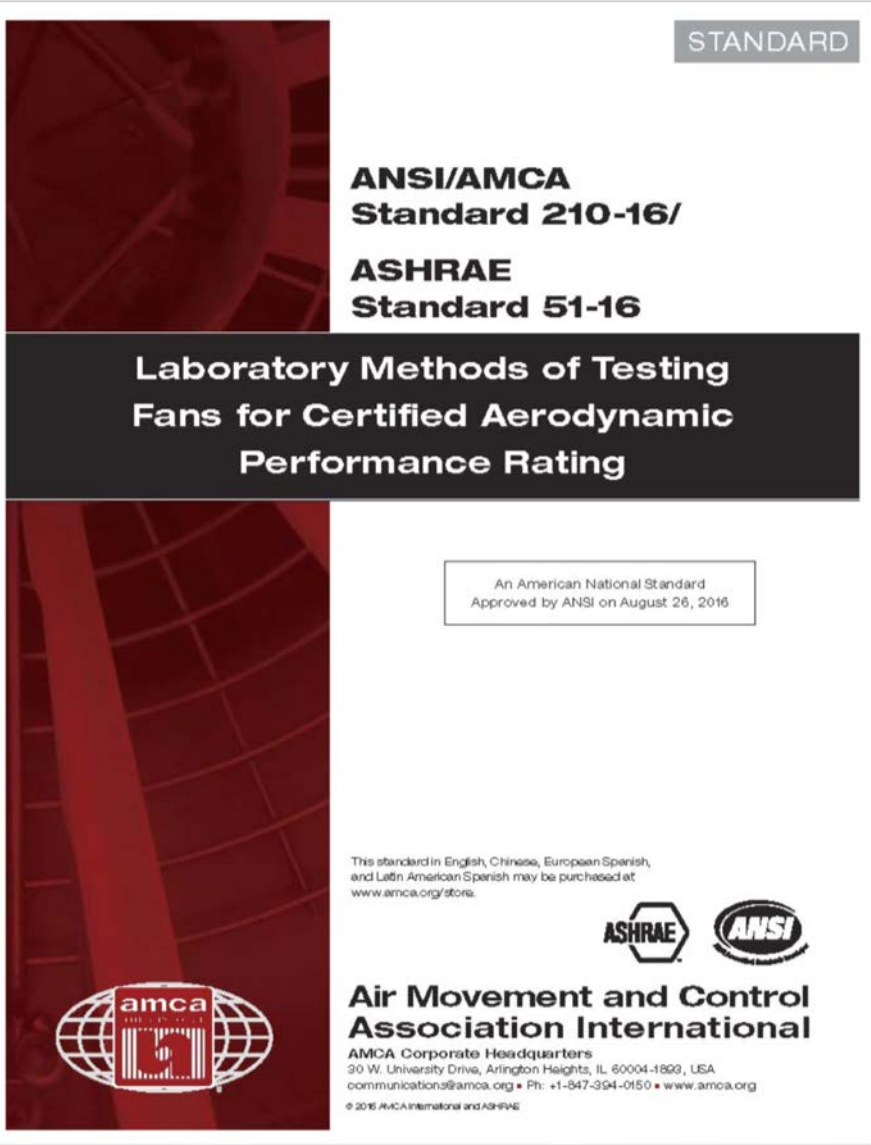
# Component-Based Fan Systems

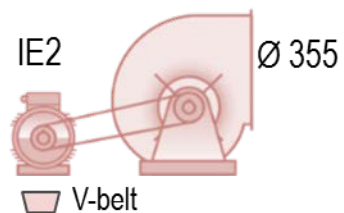
**ISO 12759:2010 Fans – Efficiency classification for fans**

"If a fan is sold as a complete assembly, **efficiency** [...] shall be **assessed by direct measurement** of  $P_e$  or  $P_{ed}$  and  $P_u$  "



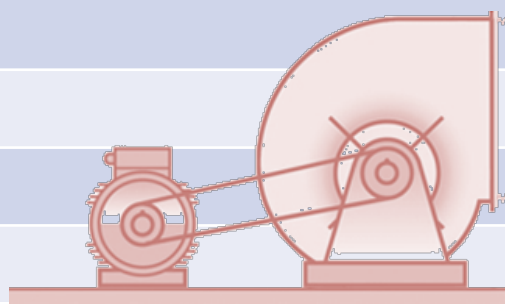
# Wire-to-air test method





Fan duty point:

$$q_V = 8,800 \text{ m}^3/\text{h} @ p_f = 1,100 \text{ Pa}$$

	AMCA 207	ISO 12759	Wire-to-air test
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Motor shaft power in kW	4.10	4.05	
Motor efficiency in %	86.5	86.6	
Wire-to-air efficiency in %	61.5	55.6 incl. $C_m$	54.5
Motor input power in kW	4.7	5.2	5.3

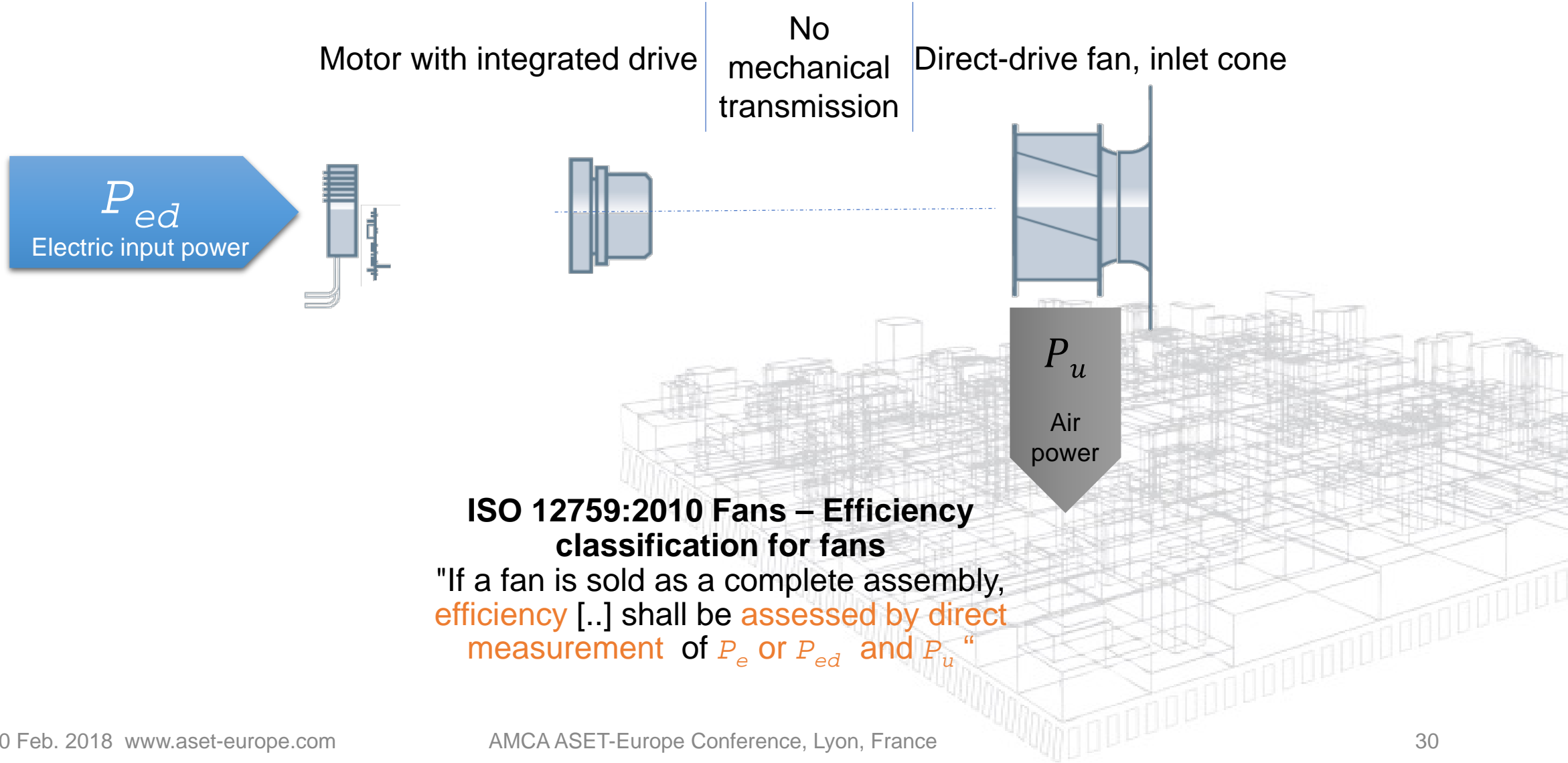
# Fully Integrated Fan System Example:

## Centrifugal fan



- Impeller, structure (housing, inlets, struts), motor, variable speed drive, electric filters
- Tested as a complete system
  - Compliant with relevant standards (motor protection, electromagnetic compatibility)

# Fully Integrated & Tested Fan Systems



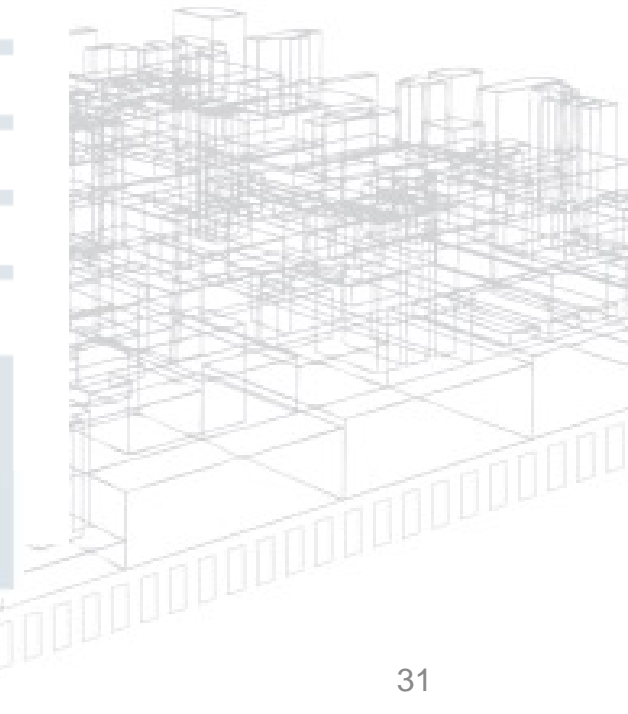
# Fully Integrated & Tested Fan Systems

Fig. 1: Accuracy class as per DIN 24166 (ISO 13348)\*

Performance characteristics	Limiting deviation in class			
	0 (AN1)	1 (AN2)	2 (AN3)	3 (AN4)
Air flow $q_v$	$\pm 1 \%$	$\pm 2.5 \%$	$\pm 5 \%$	$\pm 10 \%$
Static pressure increase $\Delta p_{\text{stat}}$	$\pm 1 \%$	$\pm 2.5 \%$	$\pm 5 \%$	$\pm 10 \%$
Drive performance $P_{\text{ed}}$	$\pm 2 \%$	$+3 \%$	$+8 \%$	$+16 \%$
Static efficiency $\eta_{\text{stat}}$	$-1 \%$	$-2 \%$	$-5 \%$	$- (-12 \%)$
Sound power level db(A)	$+3\text{dB(A)}$ $(+2\text{dB(A)})$	$+3\text{dB(A)}$	$+4\text{dB(A)}$	$+6\text{dB(A)}$
Lower power limit per DIN 24166				$> 10 \text{ kW}$

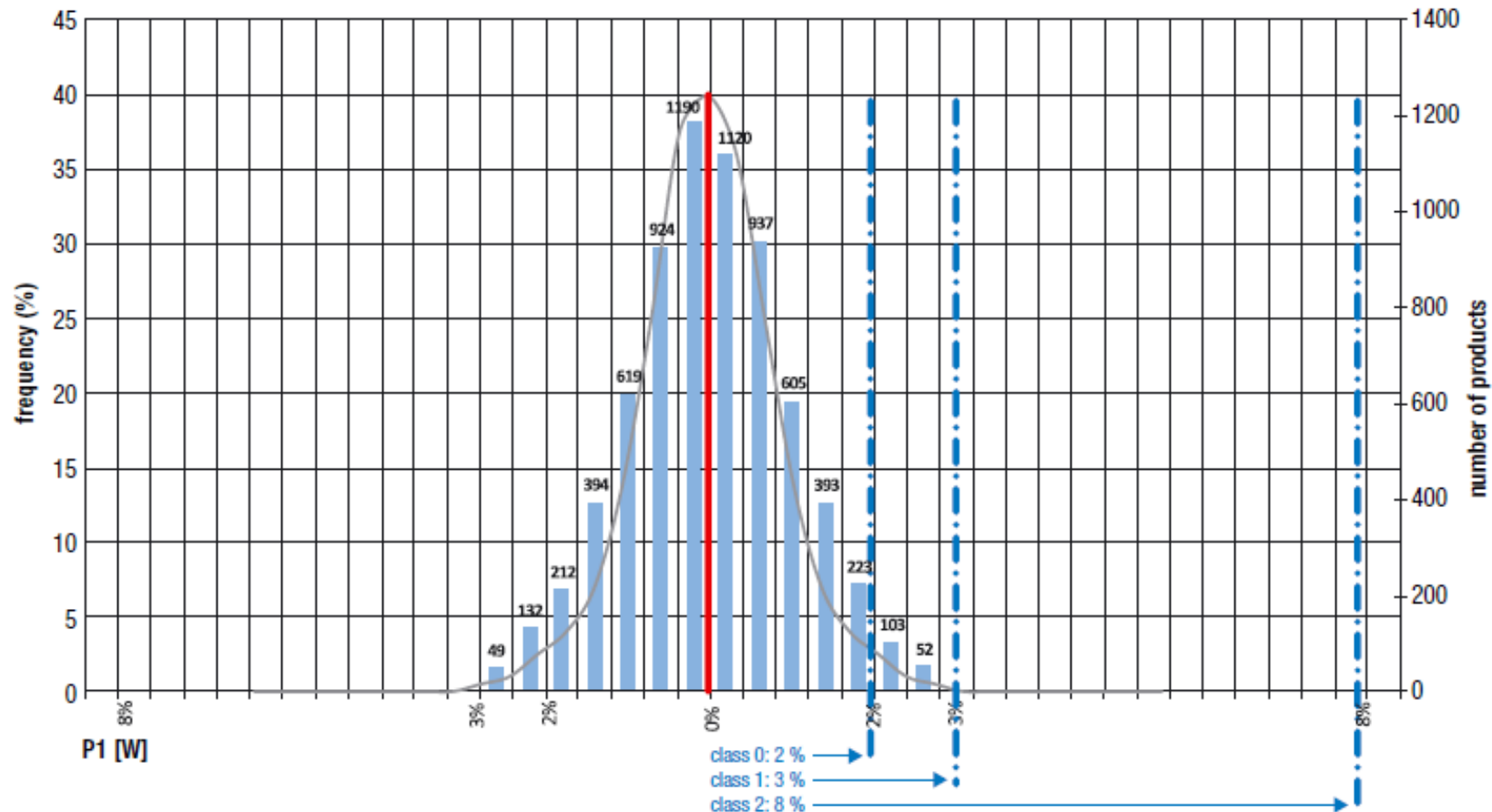
\* – ISO 13348 with other designations and slightly different values

Example:  
DIN 24166 permits  
class 3 for fans  $< 10 \text{ kW}$



# Fully Integrated & Tested Fan Systems

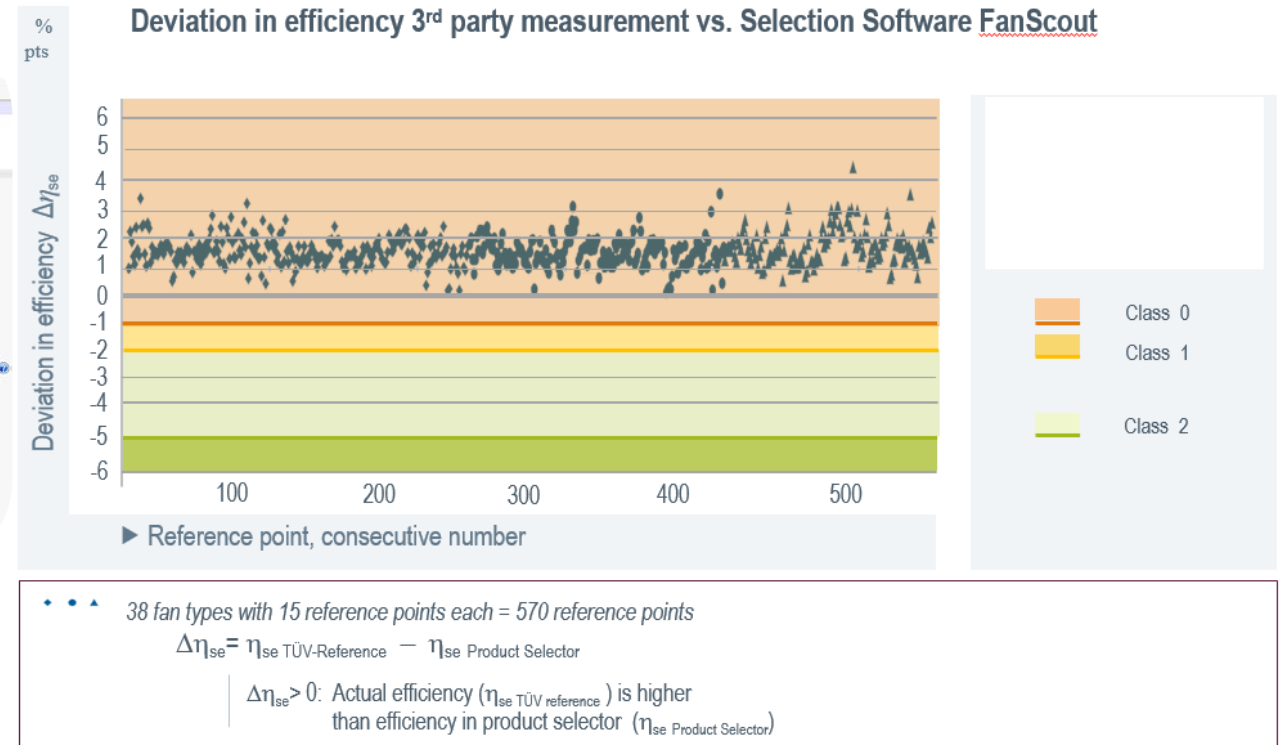
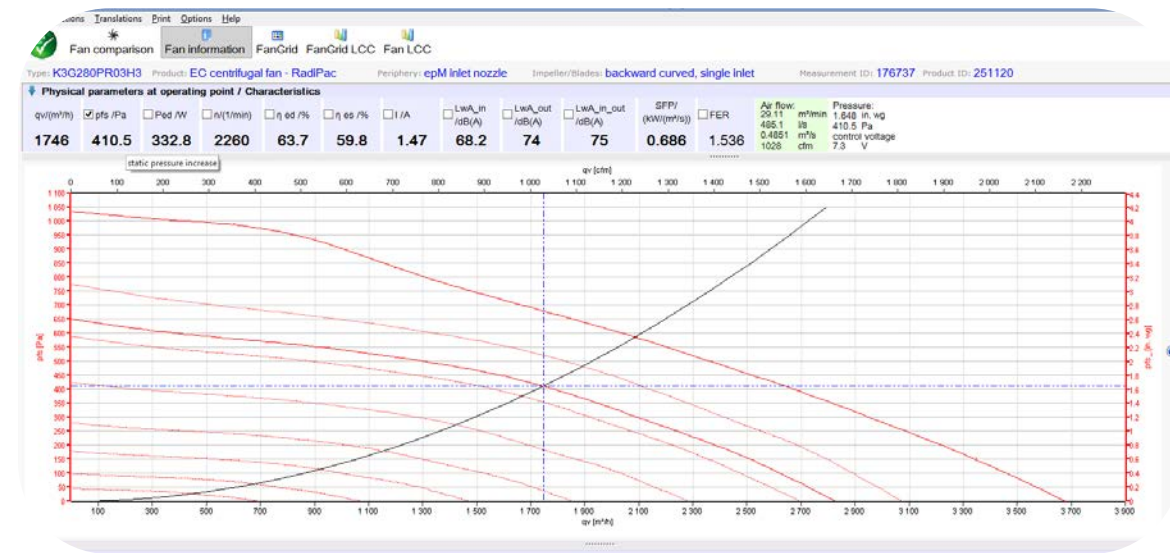
Electric input power deviation of 6953 motorized plenum fans in serial production factory.



This product range covers 1–12 kW. It meets accuracy Class 1 of DIN 24166.



# Fully Integrated & Tested Fan Systems



Fan Selection Software for AHU manufacturers accounts for fan accuracy classes

# Fully Integrated & Tested Fan Systems

Fig. 1: Accuracy class as per DIN 24166 (ISO 13348)\*

Performance characteristics	Limiting deviation in class			
	0 (AN1)	1 (AN2)	2 (AN3)	3 (AN4)
Air flow $q_v$	$\pm 1 \%$	$\pm 2.5 \%$	$\pm 5 \%$	$\pm 10 \%$
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Static efficiency $\eta_{\text{stat}}$	$-1 \%$	$-2 \%$	$-5 \%$	$- (-12 \%)$

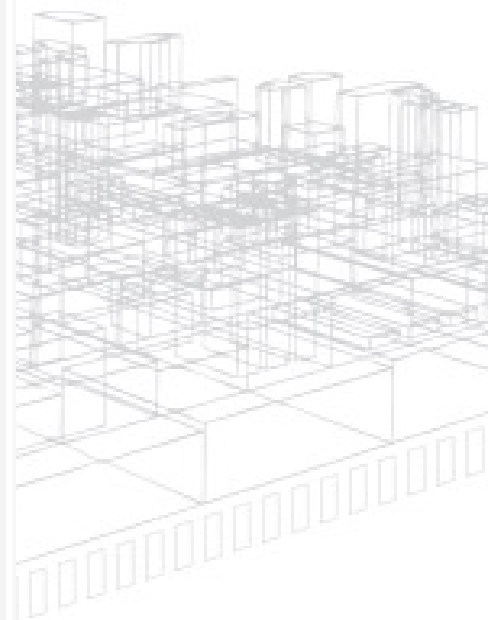
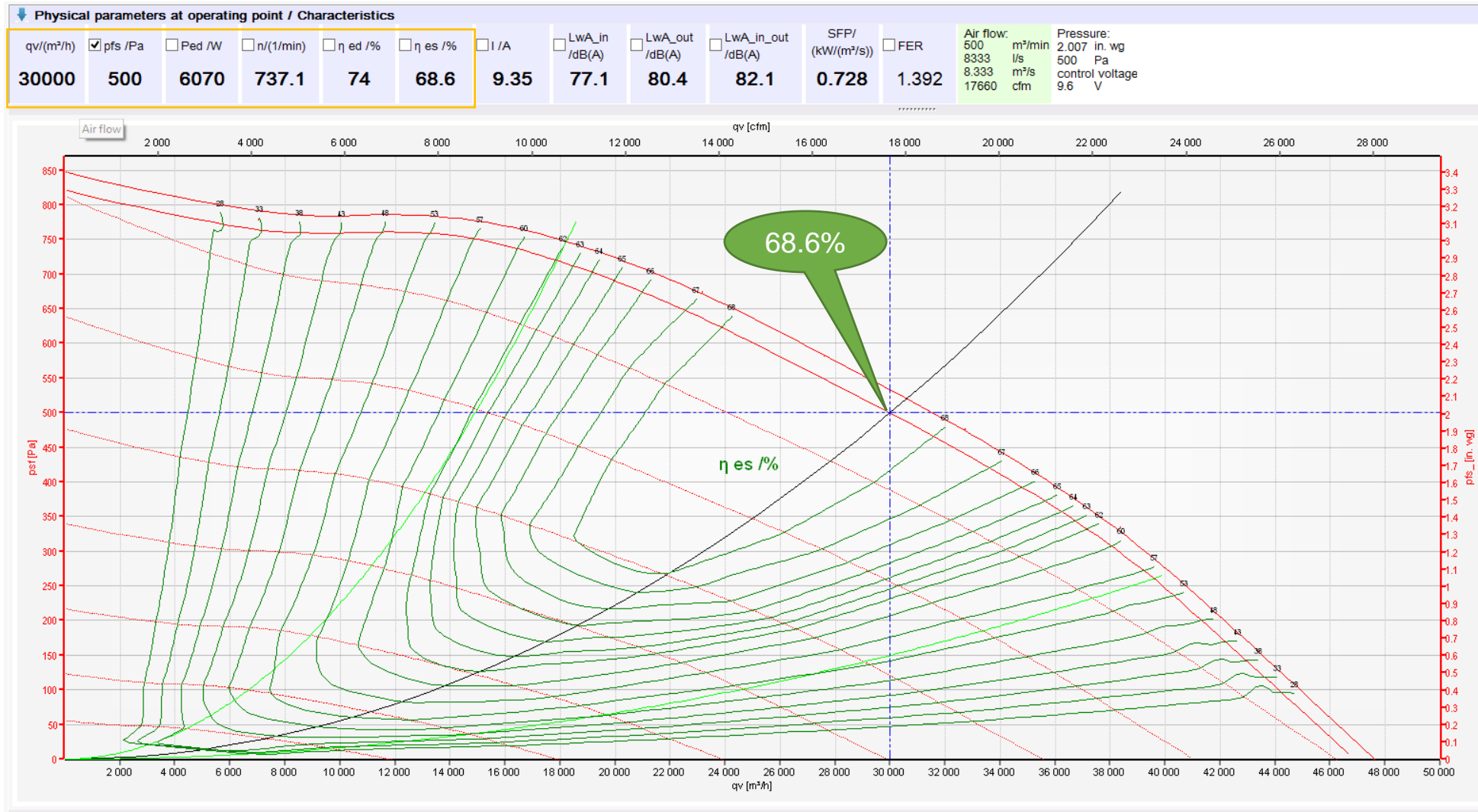
Tabelle der Berechnungsgenauigkeitsklassen

Betriebswert	Grenz-Abweichung zur Klasseneinteilung		
	B0	B1	B2
Volumenstrom	$\pm 1 \%$	$\pm 2,5 \%$	$\pm 5 \%$
Druckerhöhung	$\pm 1 \%$	$\pm 2,5 \%$	$\pm 5 \%$
Antriebsleistung	$+ 2 \%$	$+ 3 \%$	$+ 8 \%$
Wirkungsgrad	$- 1 \%$	$- 2 \%$	$- 5 \%$

Software interpolation accuracy

Production-level accuracy

# Example Specification: 30.000 m<sup>3</sup>/h at 500 Pa

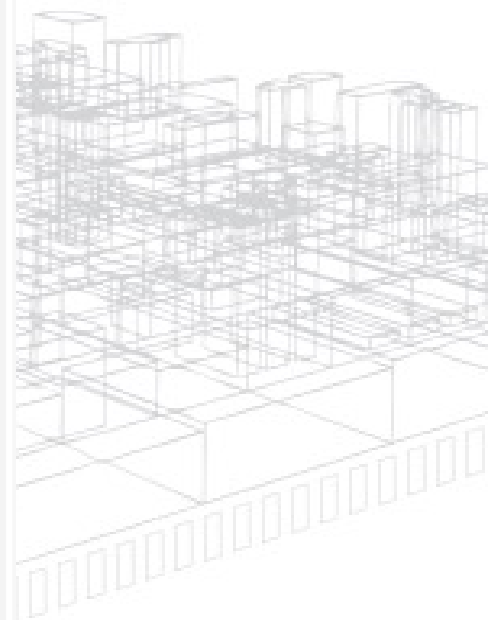
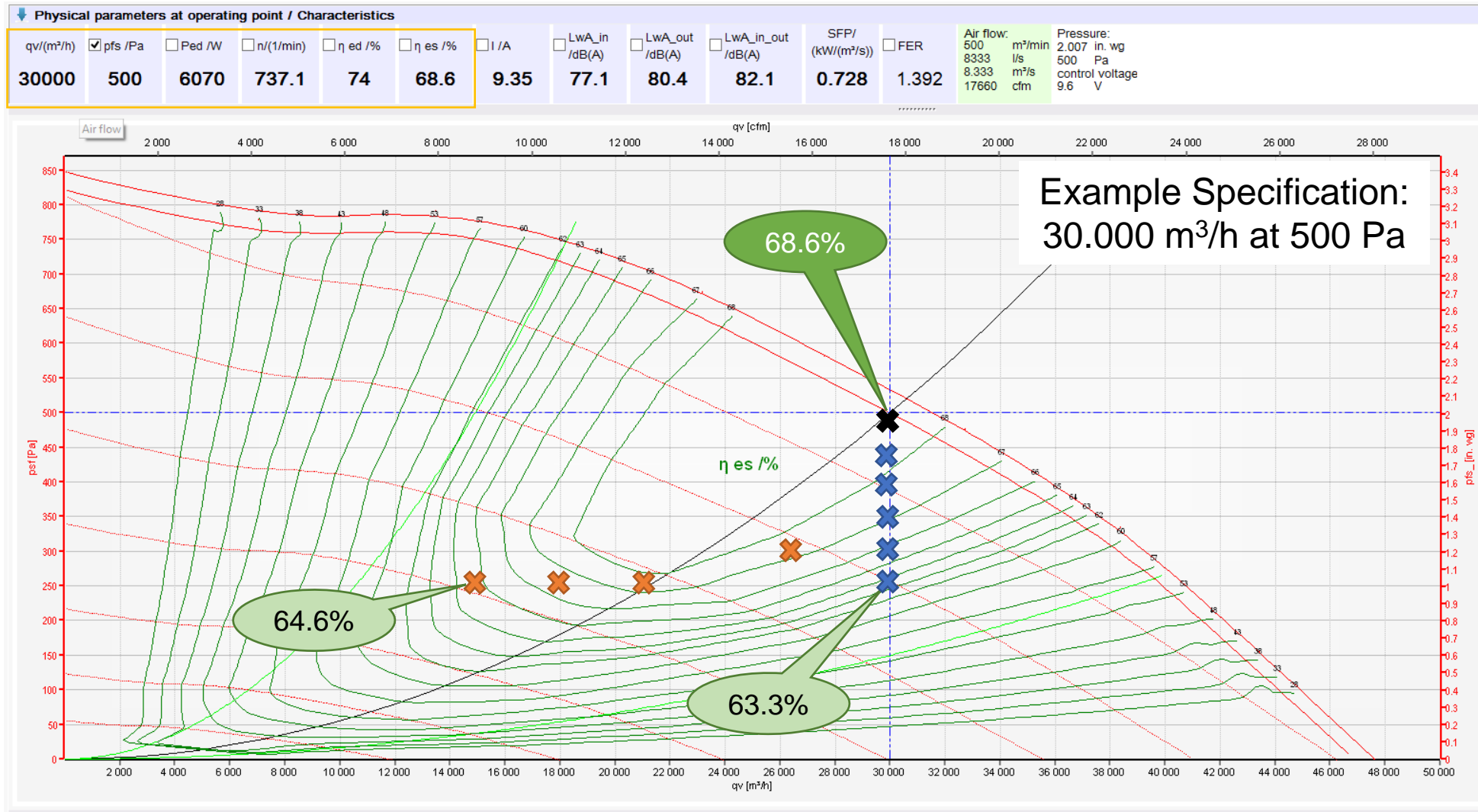


# Impact of Safety Factors on Fan Efficiency & FEI

SCENARIO 1	TSP	AIRFLOW	POWER INPUT	FEI	$\eta$	IMPACT FACTOR
DESIGN	500 Pa	30.000 m <sup>3</sup> /h	6070 W	1.58	68.6 %	-
- 10% pressure	450 Pa	30.000 m <sup>3</sup> /h	5509 W	1.60	68.1 %	99.3%
- 20% pressure	400 Pa	30.000 m <sup>3</sup> /h	4926 W	1.63	67.7 %	98.7%
- 30% pressure	350 Pa	30.000 m <sup>3</sup> /h	4370 W	1.66	66.7 %	97.2%
- 40% pressure	300 Pa	30.000 m <sup>3</sup> /h	3829 W	1.69	65.3 %	95.2%
- 50% pressure	250 Pa	30.000 m <sup>3</sup> /h	3293 W	1.73	63.3 %	92.3%

SCENARIO 2	TSP	AIRFLOW	POWER INPUT	FEI	$\eta$	IMPACT FACTOR
DESIGN	500 Pa	30.000 m <sup>3</sup> /h	6070 W	1.58	68.6 %	-
- 30 % pressure / - 10% flow	350 Pa	27.000 m <sup>3</sup> /h	3876 W	1.69	67.7 %	98.7%
- 40 % pressure / - 12% flow	300 Pa	24.000 m <sup>3</sup> /h	2966 W	1.93	67.4 %	98.3%
- 50 % pressure / -30% flow	250 Pa	21.000 m <sup>3</sup> /h	2179 W	1.87	66.9 %	97.5%
- 50 % pressure / -40% flow	250 Pa	18.000 m <sup>3</sup> /h	1904 W	1.85	65.6 %	95.6%
- 50 % pressure / -50% flow	250 Pa	15.000 m <sup>3</sup> /h	1614 W	1.85	64.6 %	94.2%

# Effects of Safety Factors on System Efficiency



# Discoveries During Retrofits

ESCO report prior to retrofitting:

DESCRIPTIONS	DETAILS of Centrifugal Fan @ Full Load	OPERATING CONDITION
LOCATION / AHU REF	AHU 2-4	
AHU MODEL		
FAN MODEL	Design	Measured
AIR FLOW	8500 m <sup>3</sup> /hr	6380 m <sup>3</sup> /hr
FAN STATIC PRESSURE (P1-P2)	880 Pa	250 Pa

Selection based on design

Type: **K3G500PB2403** Product: **EC centrifugal fan - RadiPac**

Physical parameters at operating point / Characteristics					
qv/(m <sup>3</sup> /h)	<input checked="" type="checkbox"/> pfs /Pa	<input type="checkbox"/> Ped /W	<input type="checkbox"/> n/(1/min)	<input type="checkbox"/> η ed /%	<input type="checkbox"/> η es /%
8500	880	3105	1800	70.3	66.9

Speed-controlled to actual condition

Type: **K3G500PB2403** Product: **EC centrifugal fan - RadiPac**

Physical parameters at operating point / Characteristics					
qv/(m <sup>3</sup> /h)	<input checked="" type="checkbox"/> pfs /Pa	<input type="checkbox"/> Ped /W	<input type="checkbox"/> n/(1/min)	<input type="checkbox"/> η ed /%	<input type="checkbox"/> η es /%
6380	250	779.9	1103	62.5	56.8

Input power

Efficiency drops by factor 0.85

Selection based on actual condition

Type: **K3G560PB3171** Product: **EC centrifugal fan - RadiPac**

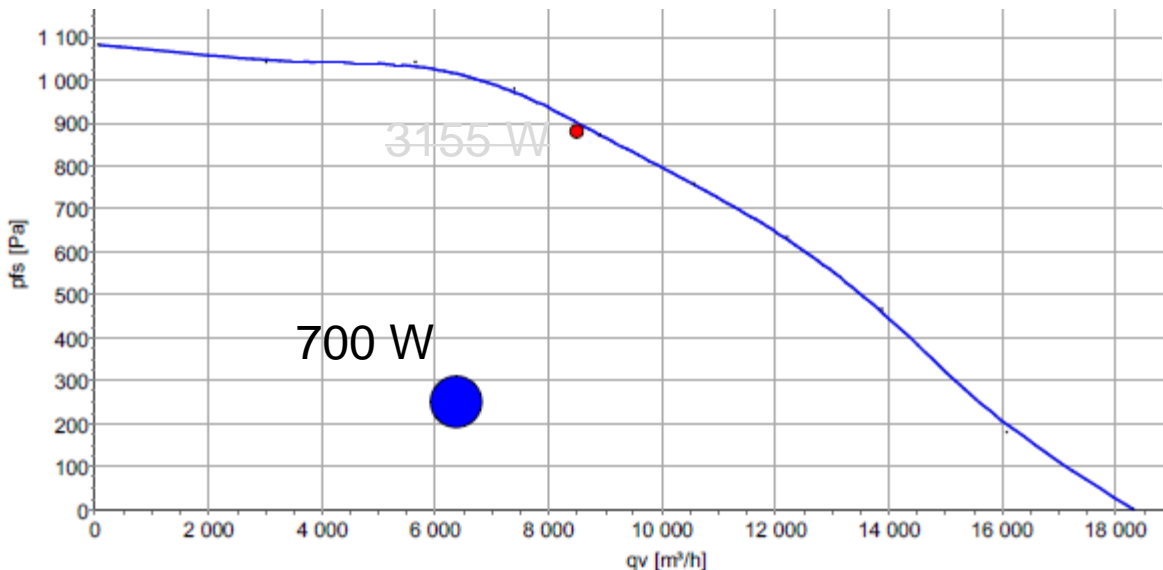
Physical parameters at operating point / Characteristics					
qv/(m <sup>3</sup> /h)	<input checked="" type="checkbox"/> pfs /Pa	<input type="checkbox"/> Ped /W	<input type="checkbox"/> n/(1/min)	<input type="checkbox"/> η ed /%	<input type="checkbox"/> η es /%
6380	250	698.5	899.6	67.6	63.4

10% input power saving through better fan sizing.



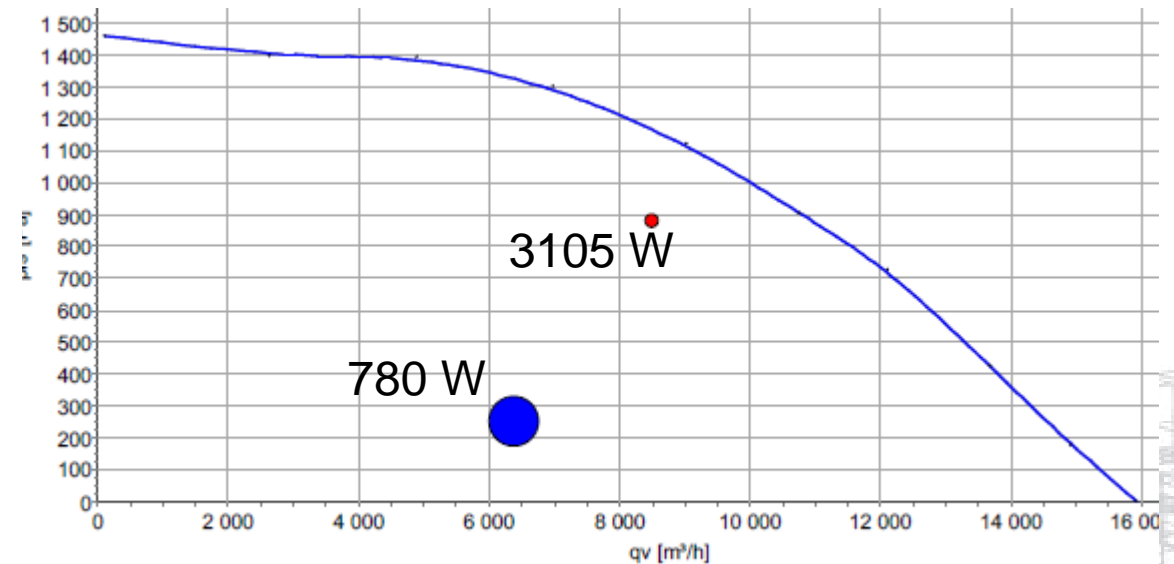
# Discoveries During Retrofits

DESCRIPTIONS	DETAILS of Centrifugal Fan @ Full Load		OPERATING CONDITION
LOCATION / AHU REF	AHU 2-4		
AHU MODEL			
FAN MODEL	Design	Measured	
AIR FLOW	8500 m <sup>3</sup> /hr	6380 m <sup>3</sup> /hr	
FAN STATIC PRESSURE (P1-P2)	880 Pa	250 Pa	



Alternate retrofit option: Fan size 560 mm

Initial retrofit option: Fan size 500 mm



- Measurements and analyses prior to retrofits provide additional saving potential through right-sizing.
- Safety factor reduction for new construction has similar effects!

# Summary

- The efficiency guarantee for highly efficient air-side equipment requires a close interaction: The mechanical designer of the building, the equipment manufacturer, and the fan system supplier.
- Fully tested fan systems with a tight accuracy class enables everyone to reduce safety factors to a minimum. Right-sized fan systems have minimal impact on design efficiency.
- For best results in retrofiting, representative measurements prior to retrofiting are recommended. The design vs. actual operation differs significantly in many cases.
- Understanding safety factors and fan system behavior supports better designs:
  - improved efficiency and modulation range → lower operational expenses
  - right-sized fan systems → lower capital expense
  - optimum return on investment



# Questions?

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